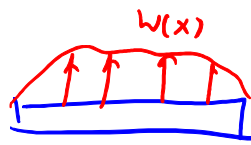


Shear-Force & Bending-Moment Diagrams

Graphical Methods



Use the relation:

$$\frac{dM}{dx} = V(x), \quad \frac{dV}{dx} = w(x)$$

Learning Objectives

- 1) To evaluate the *shear-force* and *bending-moment* diagrams for systems with discrete loads.
- 2) To do an *engineering estimate* of these quantities.

Beam Sign Convention

- Distributed load - An upward load is positive
- Shear Force - A positive internal shear force causes a clockwise rotation of beam segment. (i.e., it pushes a left-facing cross-section upward or a right-facing cross-section downward).
- Bending Moment - A positive internal moment causes compression in the top fibers of the segment (i.e., clockwise on a left-facing cross-section or counter-clockwise on a right-facing cross-section).

Procedure

1. Determine support reactions
2. Specify beam sections origin (left end) to between each discrete load (force or moment). Be sure V and M are shown acting in the positive sense.
3. Sum forces vertically to determine V
4. Sum moments at sectioned end to determine M . (This eliminates V from the moment equation).

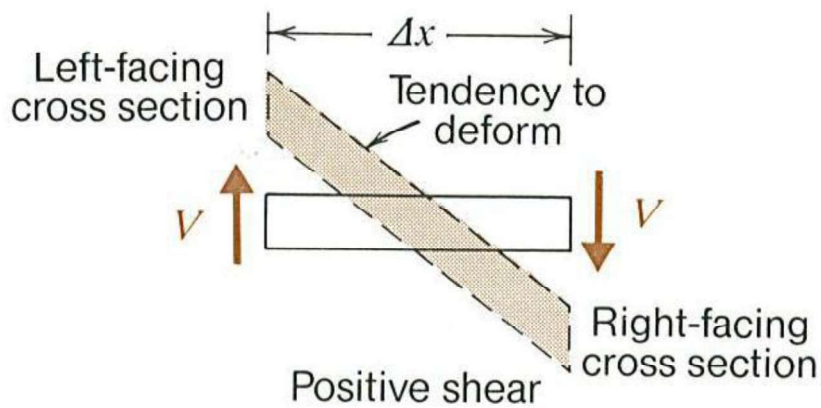
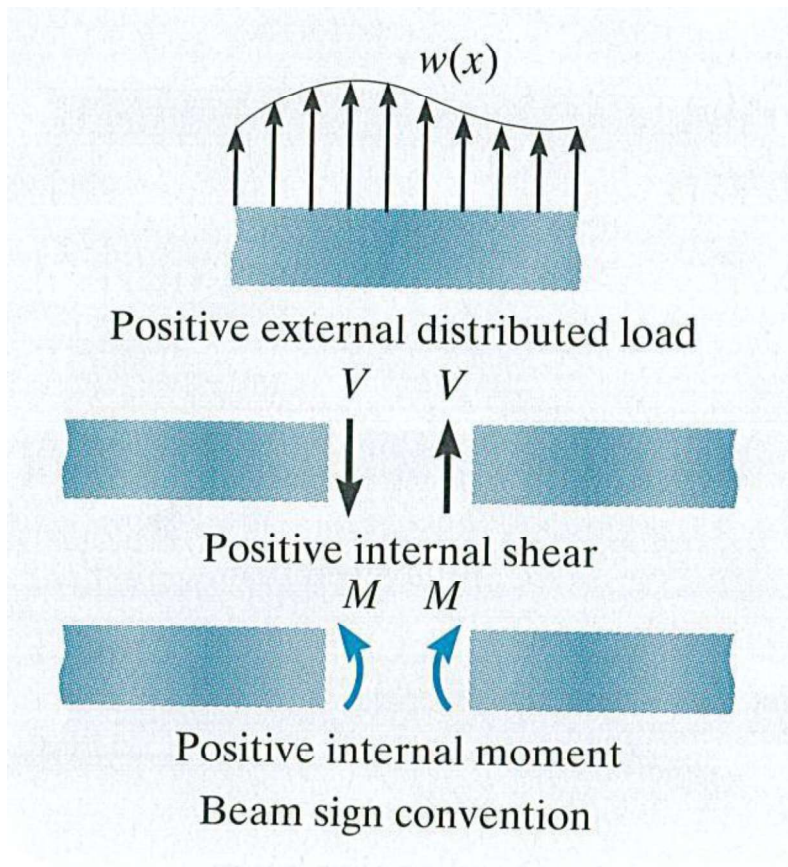


FIGURE 7a

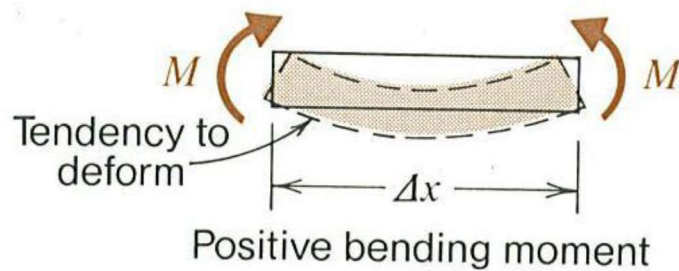
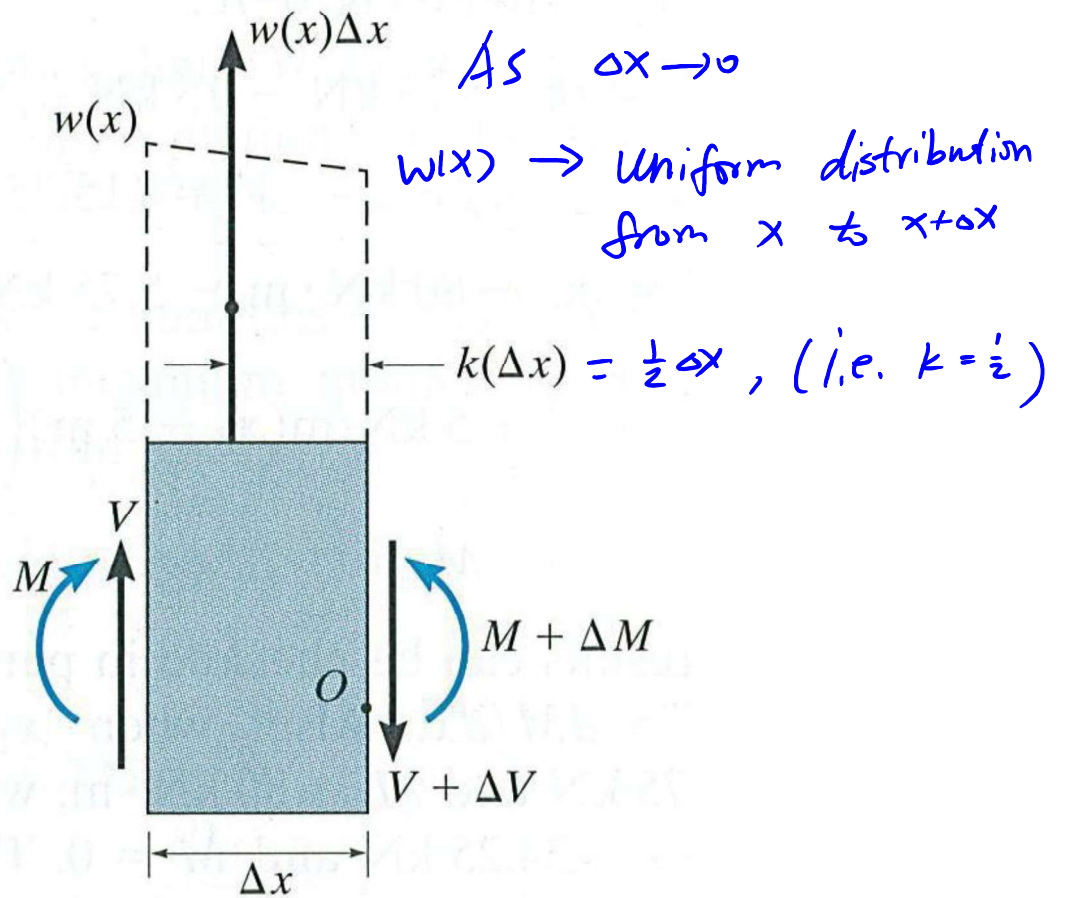
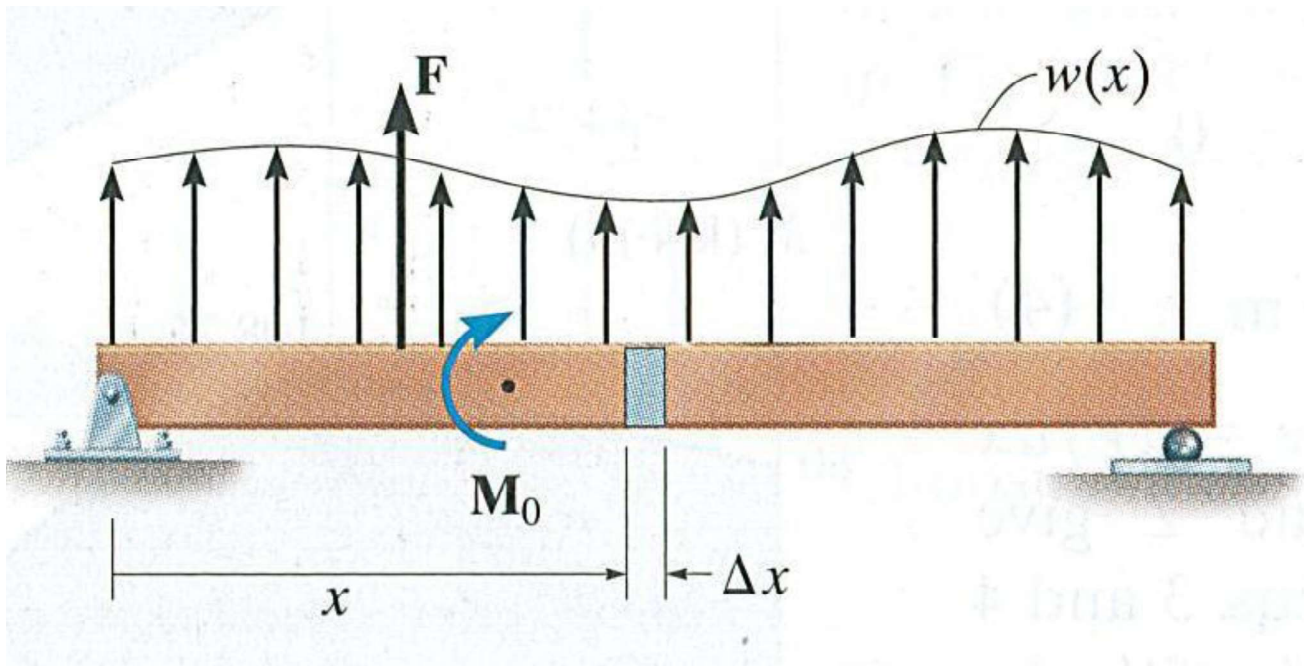


FIGURE 7b



Free-body diagram of segment Δx

(b)

$$\sum \vec{F}_y = 0: V + W(x)\Delta x - (V + \Delta V) = 0$$

$$\Rightarrow \frac{\Delta V}{\Delta x} = W(x)$$

$$\lim_{\Delta x \rightarrow 0} \frac{\Delta V}{\Delta x} = \frac{dV}{dx} = W(x)$$

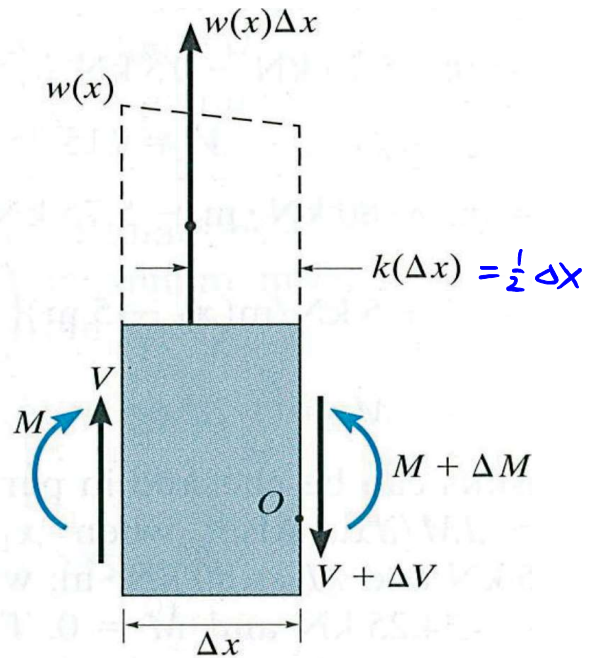
$$\sum M_o = 0:$$

$$M + \Delta M - M - V \cdot \Delta x - (W(x)\Delta x)\left(\frac{1}{2}\Delta x\right) = 0$$

$$\Rightarrow \frac{\Delta M}{\Delta x} = V + \frac{1}{2}W(x)\Delta x$$

$\searrow 0$ as $\Delta x \rightarrow 0$

$$\lim_{\Delta x \rightarrow 0} \frac{\Delta M}{\Delta x} = \frac{dM}{dx} = V$$

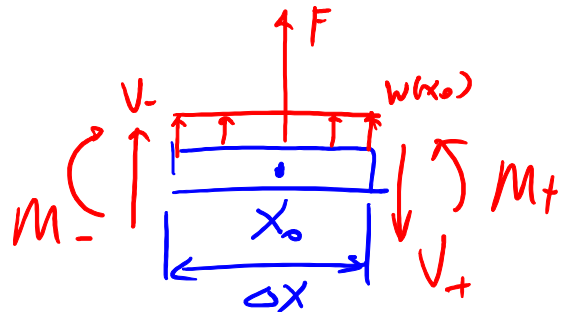


- Concentrated Load: F at x_0

$$\sum \vec{F}_y = 0: V_- - V_+ + F + \frac{W(x_0)\Delta x}{\Delta x} = 0$$

$\searrow 0$ as $\Delta x \rightarrow 0$

$$\Delta x \rightarrow 0 \Rightarrow V_+ - V_- = F$$

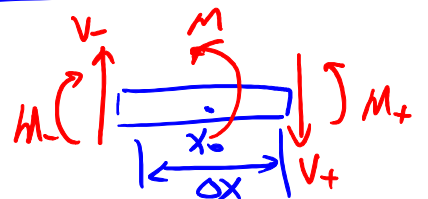


$$\sum M_{x_0} = 0: -M_- + M_+ - V_- \left(\frac{1}{2}\Delta x\right) - V_+ \left(\frac{1}{2}\Delta x\right) = 0$$

$$\Delta x \rightarrow 0 \Rightarrow M_- = M_+$$

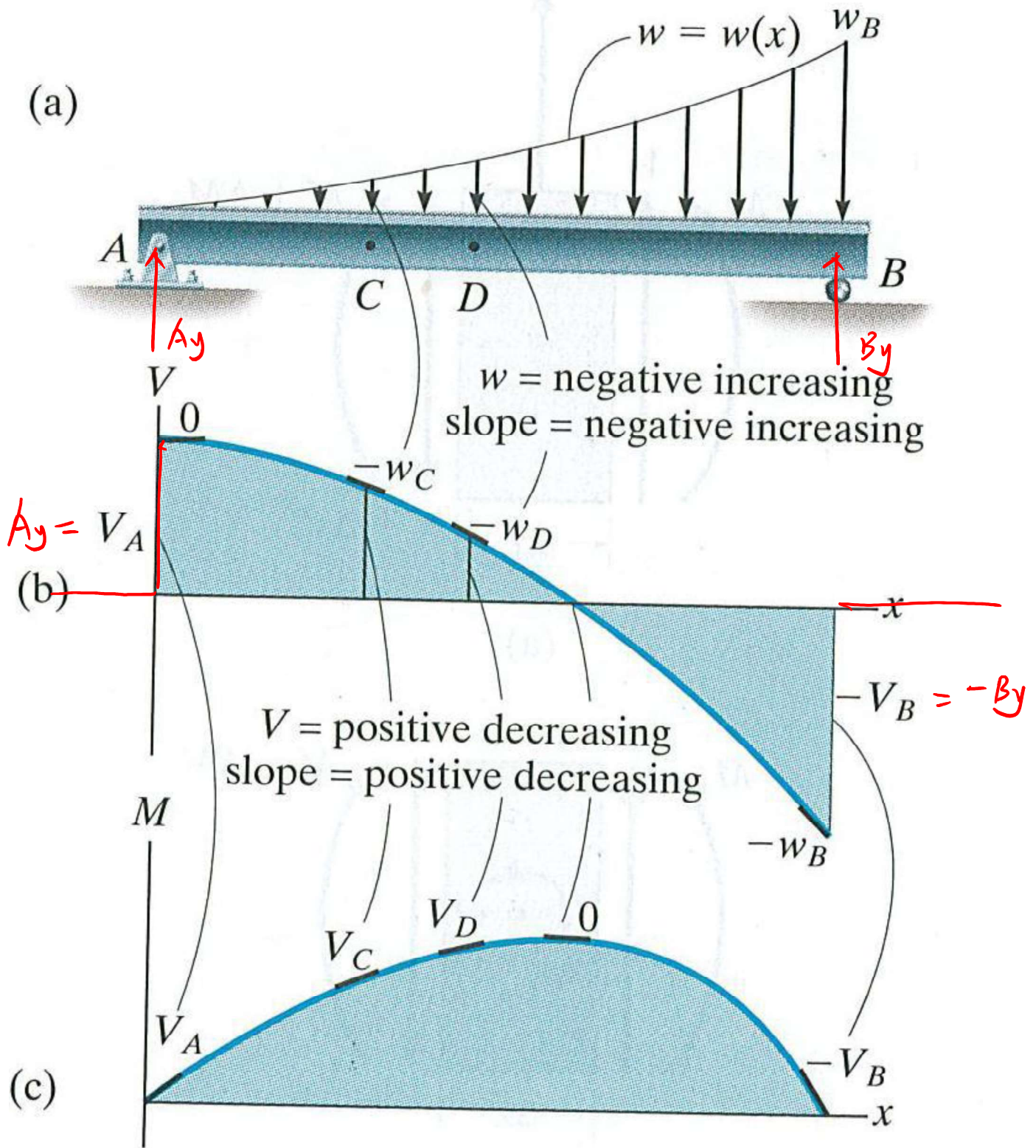
- Concentrated Moment: M at x_0

$$\sum \vec{F}_y = 0: V_- = V_+$$



$$\sum M_{x_0} = 0: M_+ - M_- + M - V_- \left(\frac{1}{2}\Delta x\right) - V_+ \left(\frac{1}{2}\Delta x\right) = 0$$

$$\Delta x \rightarrow 0 \Rightarrow M_+ - M_- = -M$$

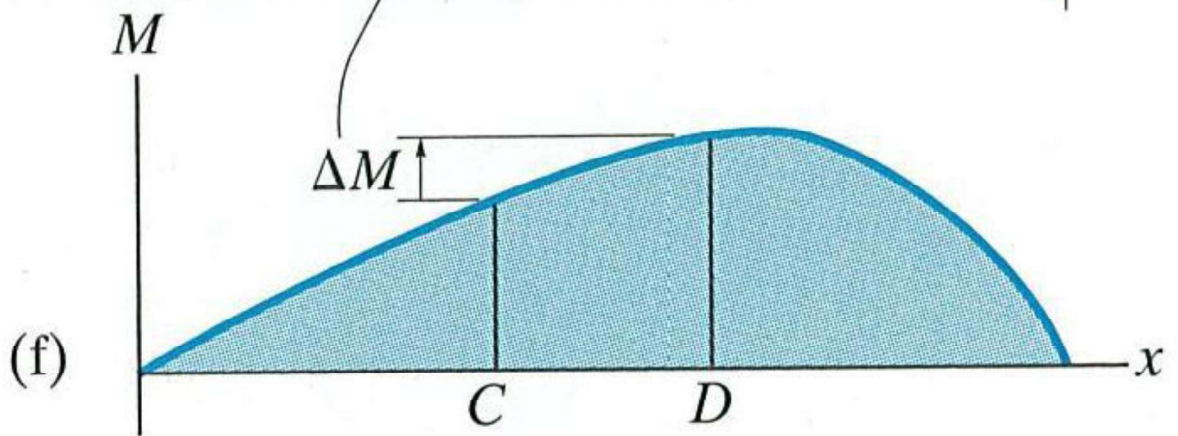
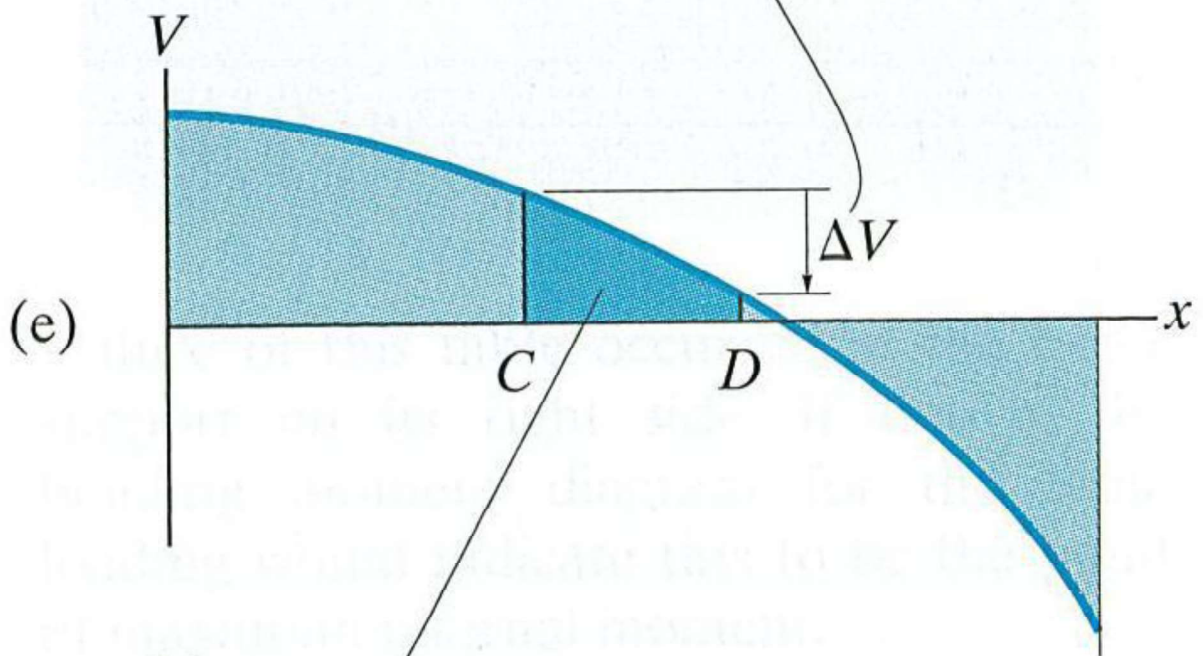
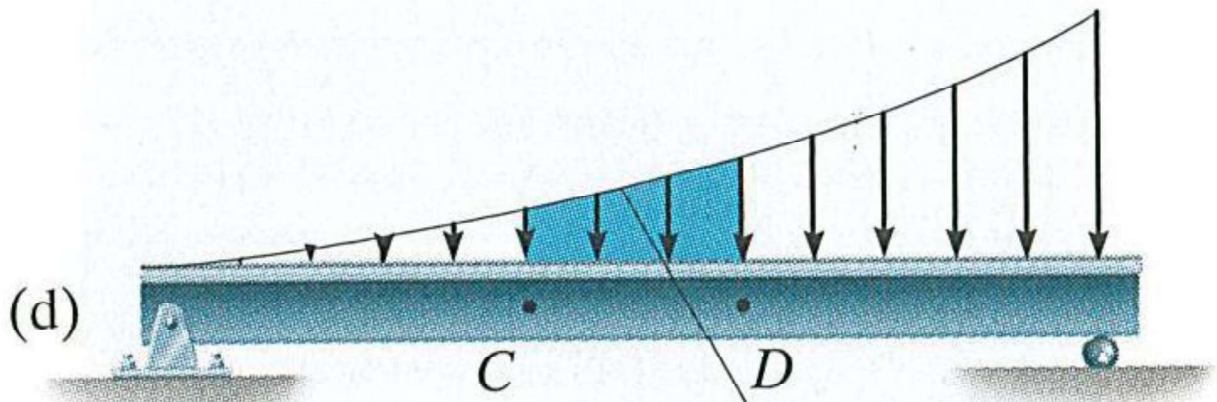


$$\Delta V = \int w(x) dx$$

change in shear = area under distributed loading

$$\Delta M = \int V(x) dx$$

change in moment = area under shear diagram



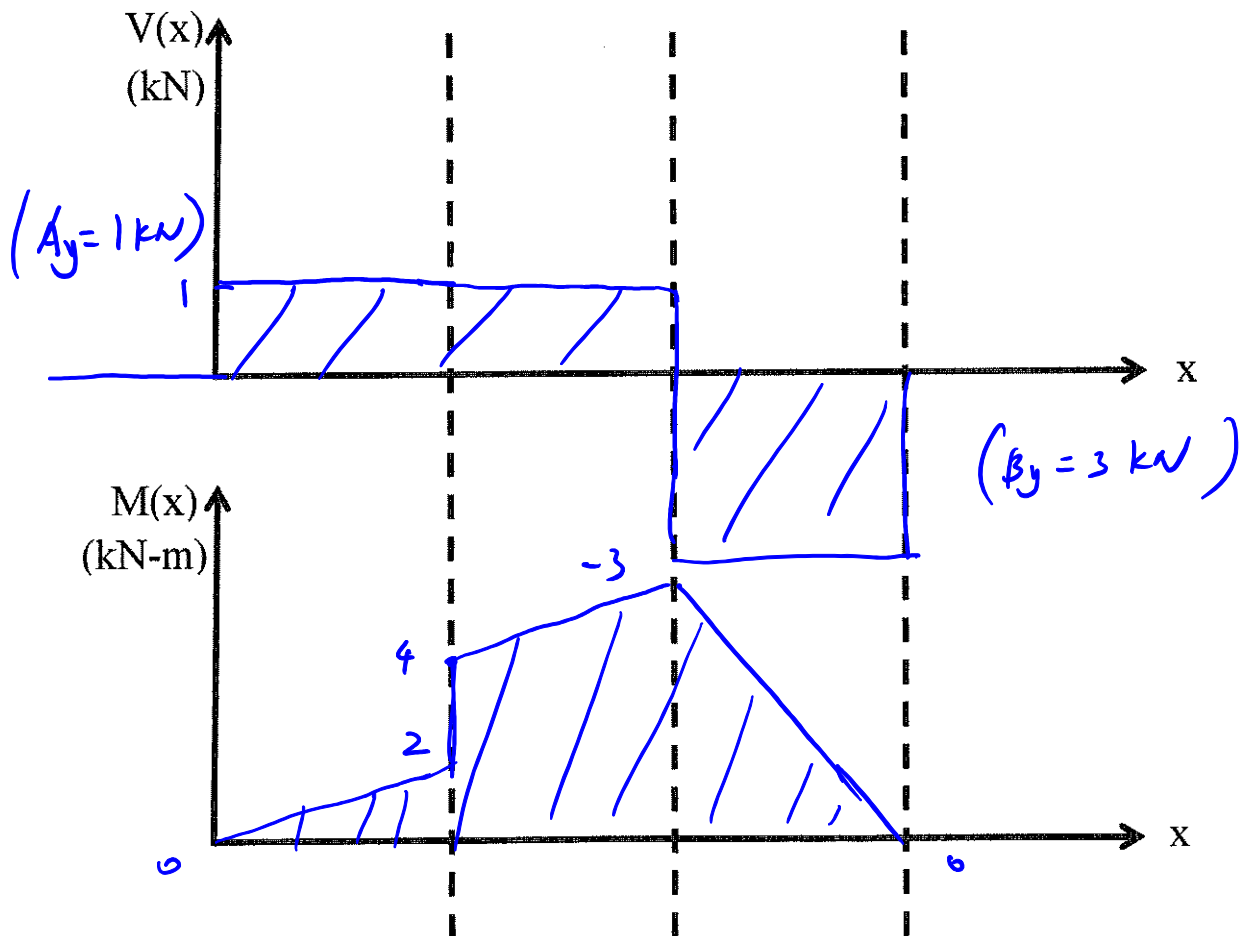
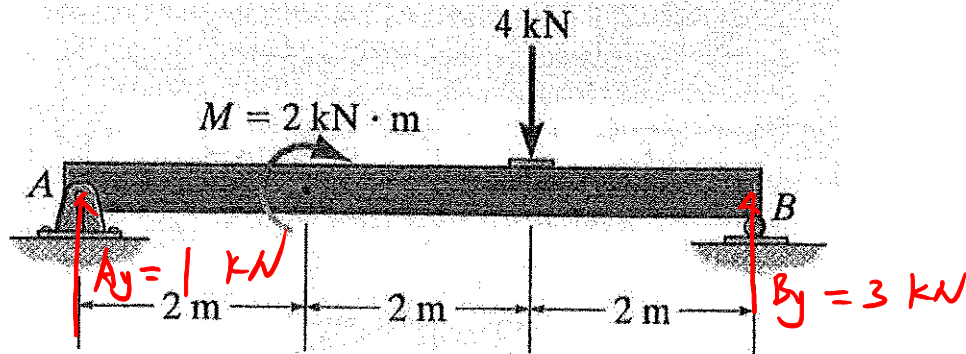
Shear-Force & Bending-Moment Diagrams

Graphical Methods

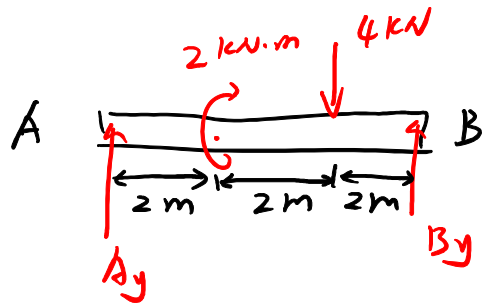
Example 1

Given: A simply-supported beam is loaded with a 2 kN-m couple and a 4 kN load as shown.

Find: Using graphical methods, draw the shear-force and bending-moment diagrams.



Reaction:



$$\sum M_A = 0 : -2 - 4(4) + B_y(6) = 0$$

$$\Rightarrow B_y = 3 \text{ kN}$$

$$\sum F_y = 0 : A_y + B_y - 4 = 0$$

$$\Rightarrow A_y = 1 \text{ kN}$$

Shear-Force & Bending-Moment Diagrams

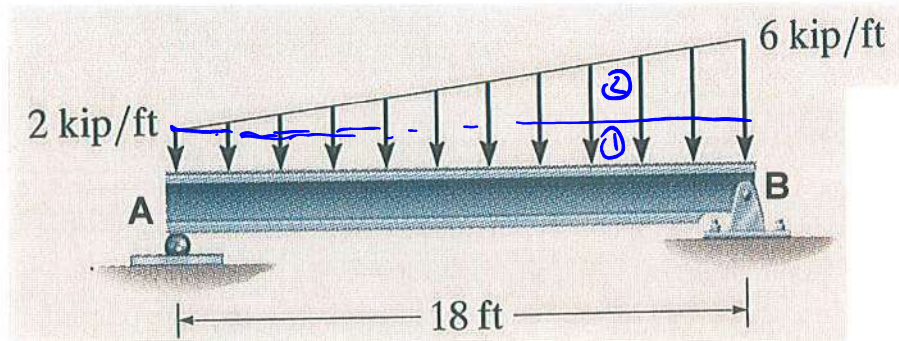
Graphical Methods

Example 2

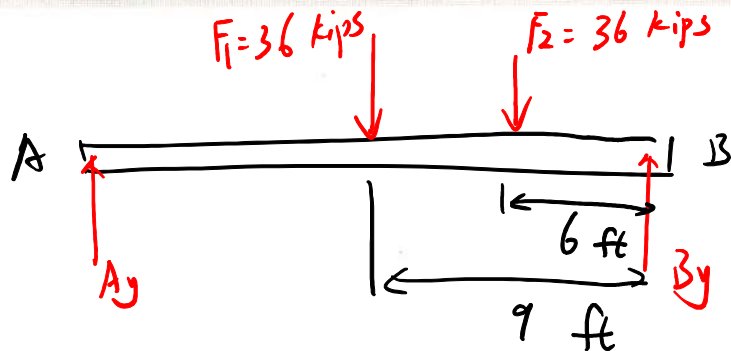
Given: A simply supported beam is supporting a trapezoidal loading as shown and is in static equilibrium.

Find:

- Determine the reactions at supports A and B.
- Write an algebraic expression for the trapezoidal loading shown ($w(x)$).
- Using the relationship $\frac{dV}{dx} = w(x)$, determine an expression for the shear force as a function of x (i.e., $V(x)$).
- Using the relationship $\frac{dM}{dx} = V(x)$, determine an expression for the bending moment as a function of x (i.e., $M(x)$).



a) Reaction:



$$\sum M_A = 0: -F_1 \cdot (9) - F_2 \cdot (12) + B_y \cdot (18) = 0$$

$$\Rightarrow B_y = 42 \text{ kips}$$

$$\sum F_y = 0: A_y + B_y - F_1 - F_2 = 0$$

$$\Rightarrow A_y = 30 \text{ kips}$$

$$b). \quad w(x) = -2 - \frac{2}{9}x \quad \text{kip/ft}$$

$$c) \quad \frac{dV}{dx} = w(x) = -\frac{2}{9}x - 2$$

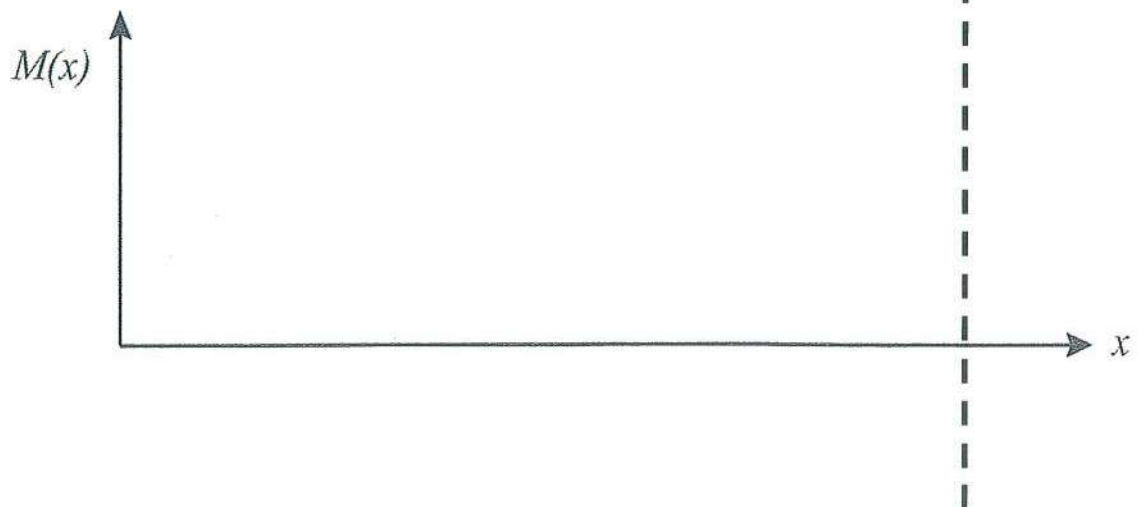
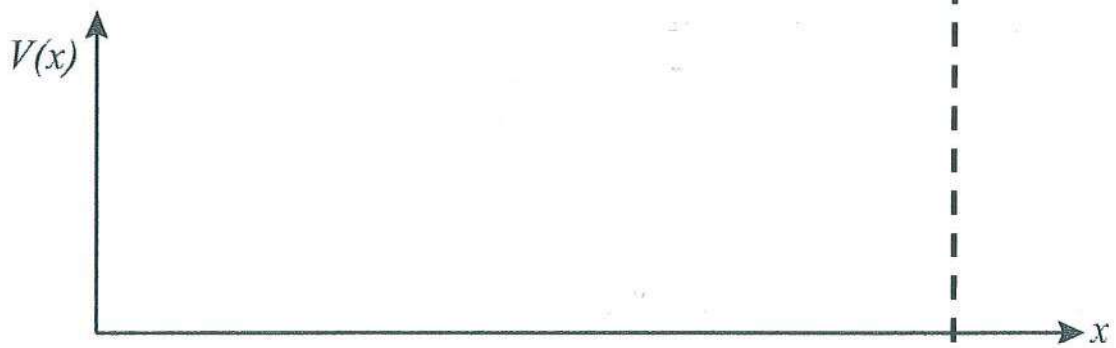
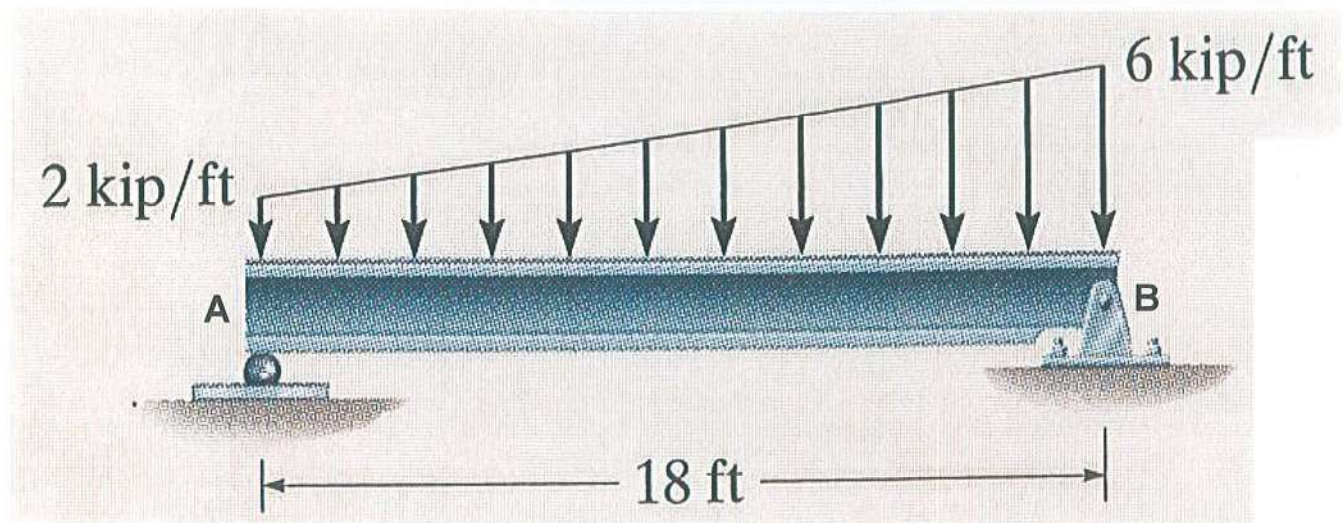
$$V(x) - \underbrace{V(0)}_{Ay=30} = \int_0^x w(s) ds = \int_0^x \left(-\frac{2}{9}s - 2\right) ds \\ = \left[-\frac{1}{9}s^2 - 2s\right] \Big|_0^x = -\frac{1}{9}x^2 - 2x$$

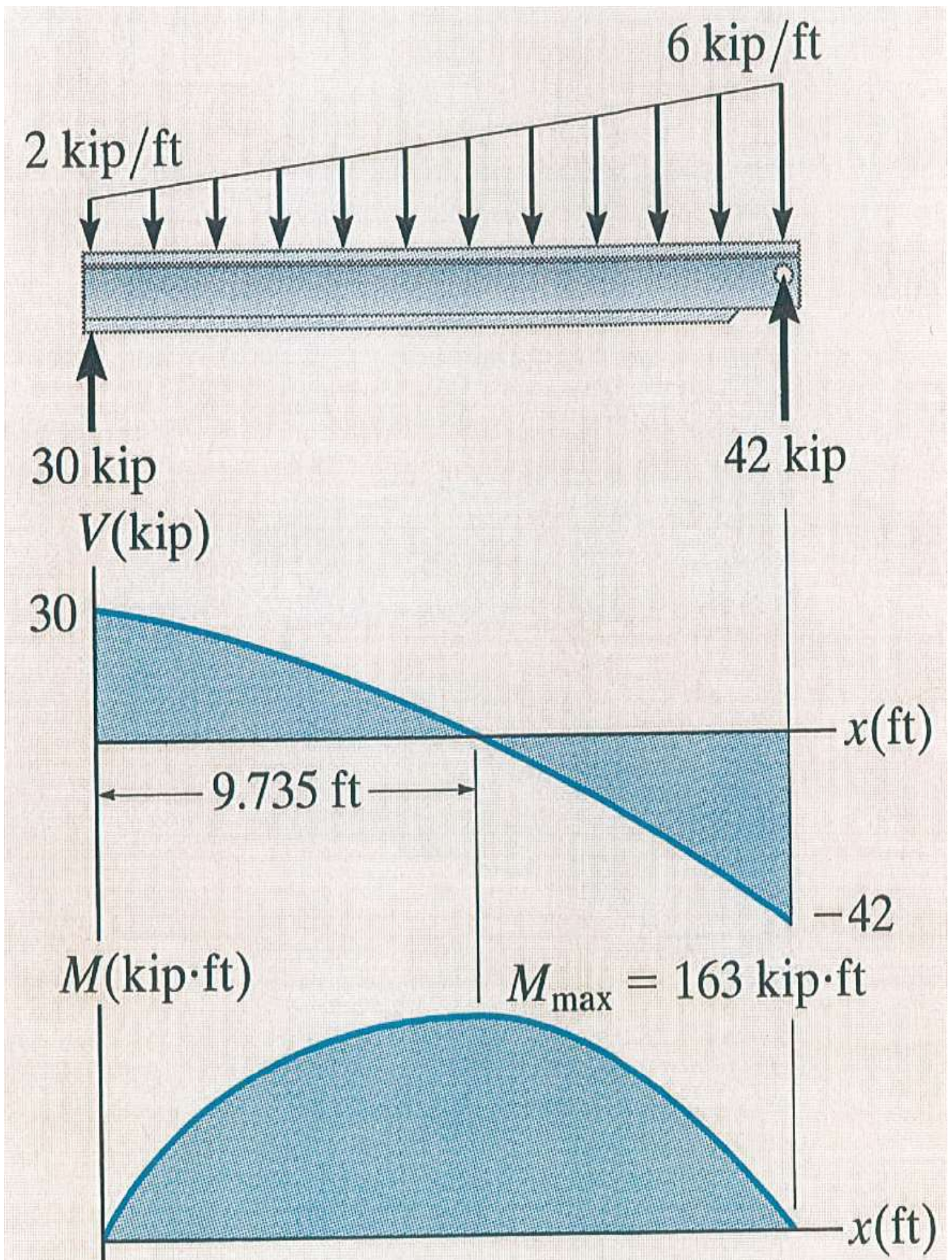
$$\Rightarrow V(x) = -\frac{x^2}{9} - 2x + 30 \quad \text{kips}$$

$$d) \quad \frac{dM}{dx} = V(x) = -\frac{x^2}{9} - 2x + 30$$

$$M(x) - \underbrace{M(0)}_0 = \int_0^x V(s) ds = \int_0^x \left(-\frac{s^2}{9} - 2s + 30\right) ds \\ = \left[-\frac{s^3}{27} - s^2 + 30s\right] \Big|_0^x \\ = -\frac{x^3}{27} - x^2 + 30x$$

$$M(x) = -\frac{x^3}{27} - x^2 + 30x \quad \text{kip}\cdot\text{ft.}$$



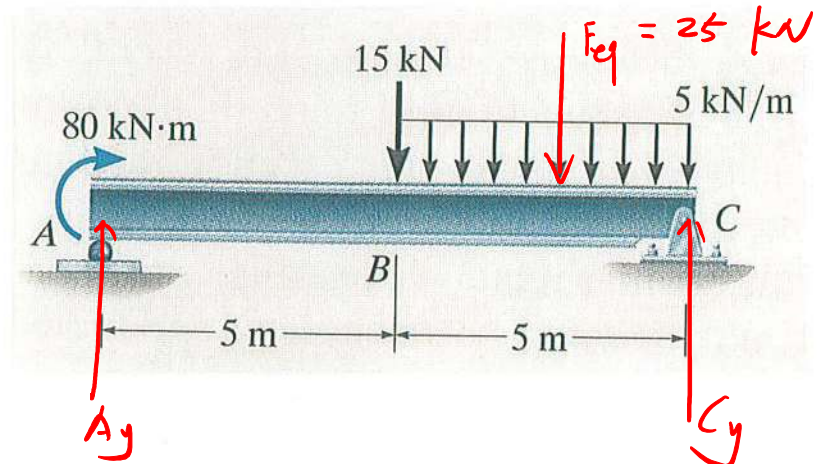


Shear-Force and Bending-Moment Diagrams

Example 3

Given: A simply-supported beam is loaded as shown.

Find: Using graphical methods, draw the shear-force and bending-moment diagrams on the axes provided below.

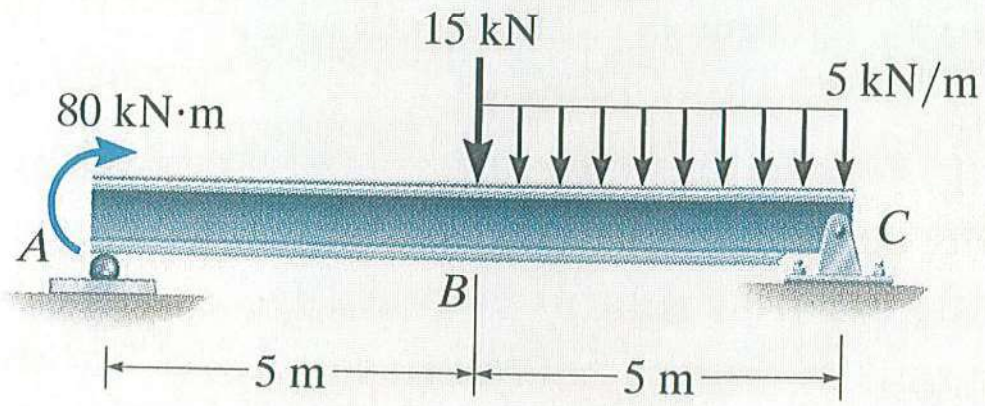


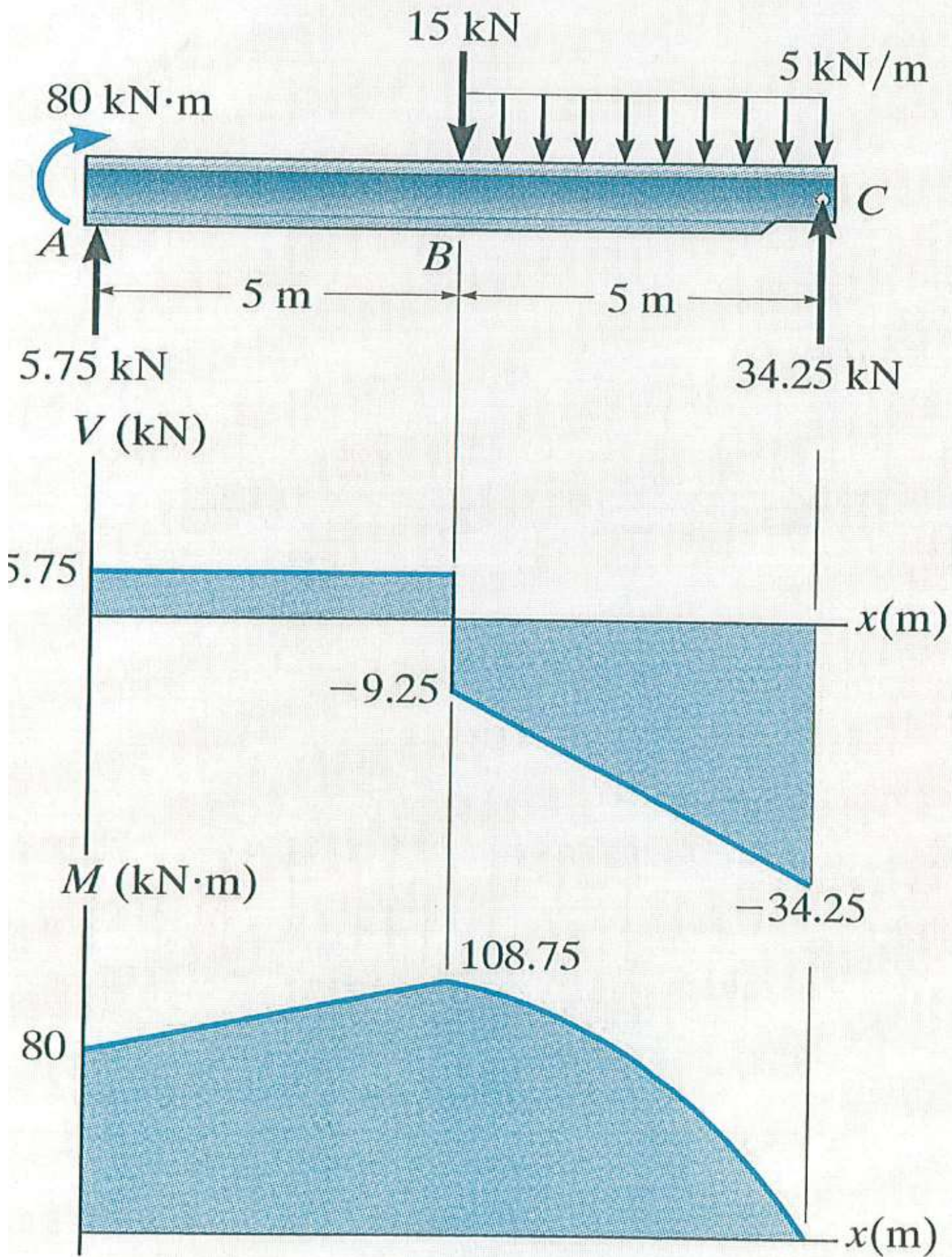
$$\sum M_C = 0: \quad -80 - A_y(10) + 15(5) + 25(2.5) = 0$$

$$\Rightarrow A_y = 5.75 \text{ kN}$$

$$\sum F_y = 0: \quad A_y - 15 - 25 + C_y = 0$$

$$\Rightarrow C_y = 34.25 \text{ kN}$$



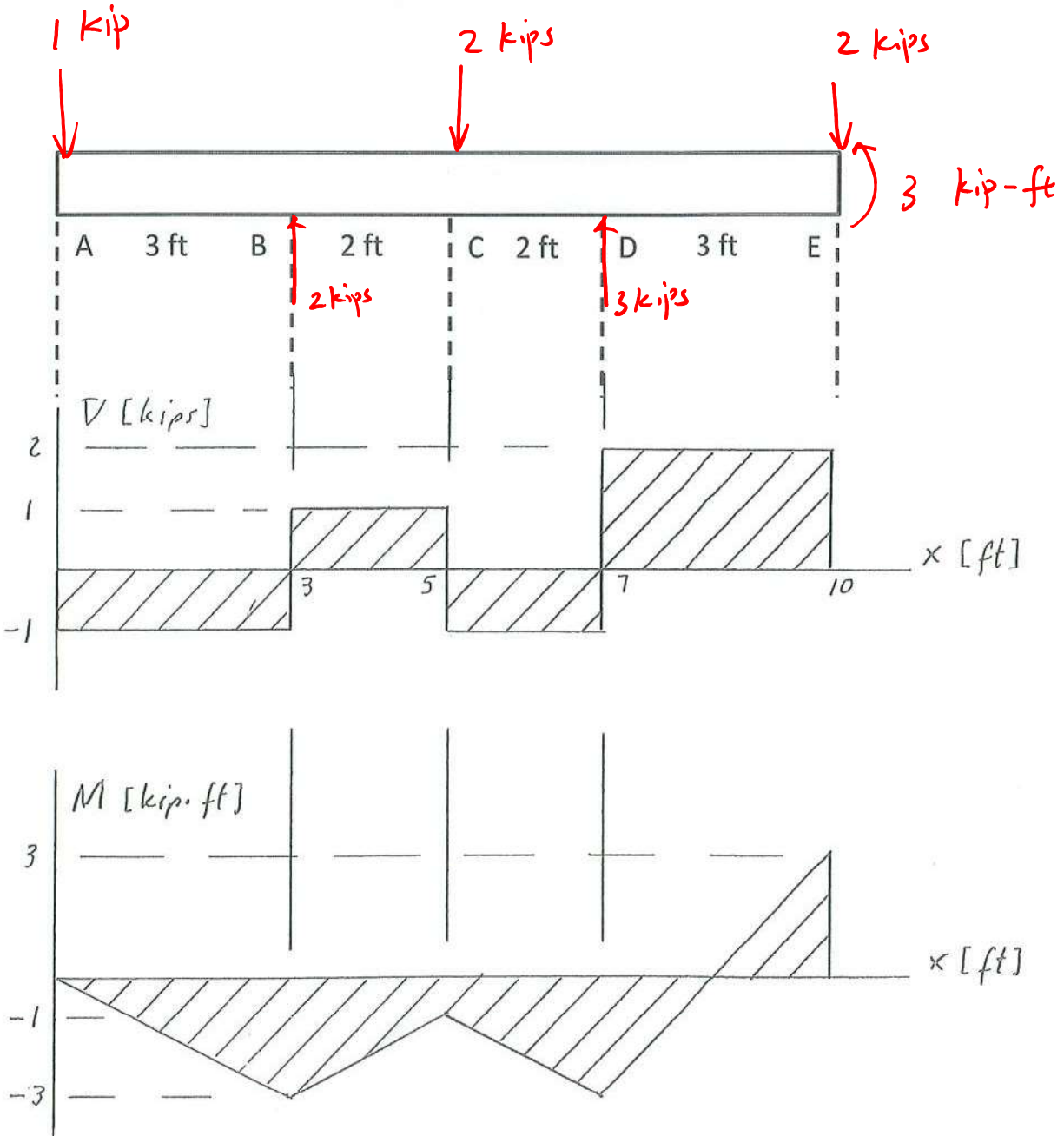


Shear-Force and Bending-Moment Diagrams

Example 4

Given: The shear-force and bending-moment diagrams for a loaded beam are provided below.

Find: On the beam provided, sketch a valid set of loads that would result in the shear-force and bending-moment diagrams shown.



ME 270 – Basic Mechanics I – Group Quiz

Your Name: _____ Group Members: 1) _____

Date: _____ Period: _____ 2) _____

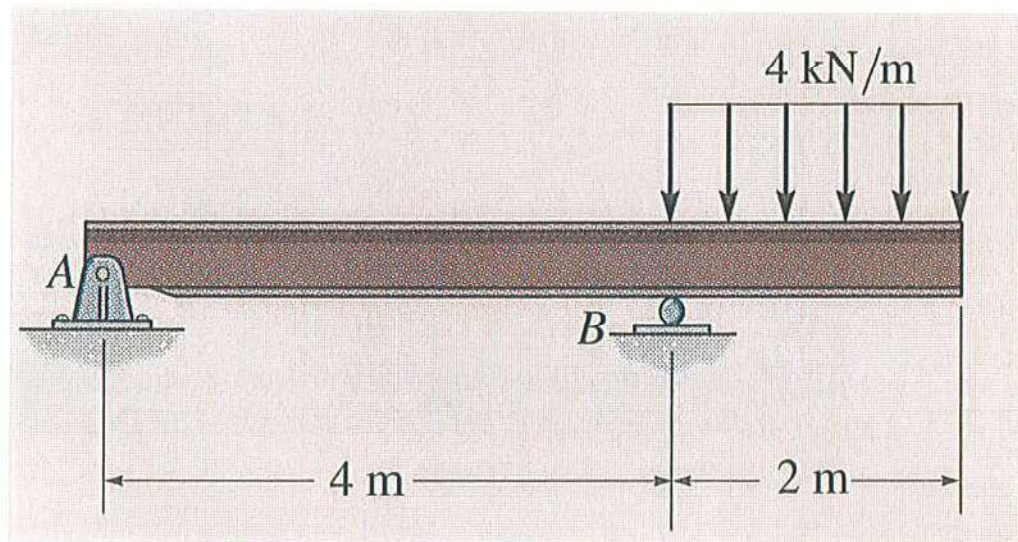
3) _____

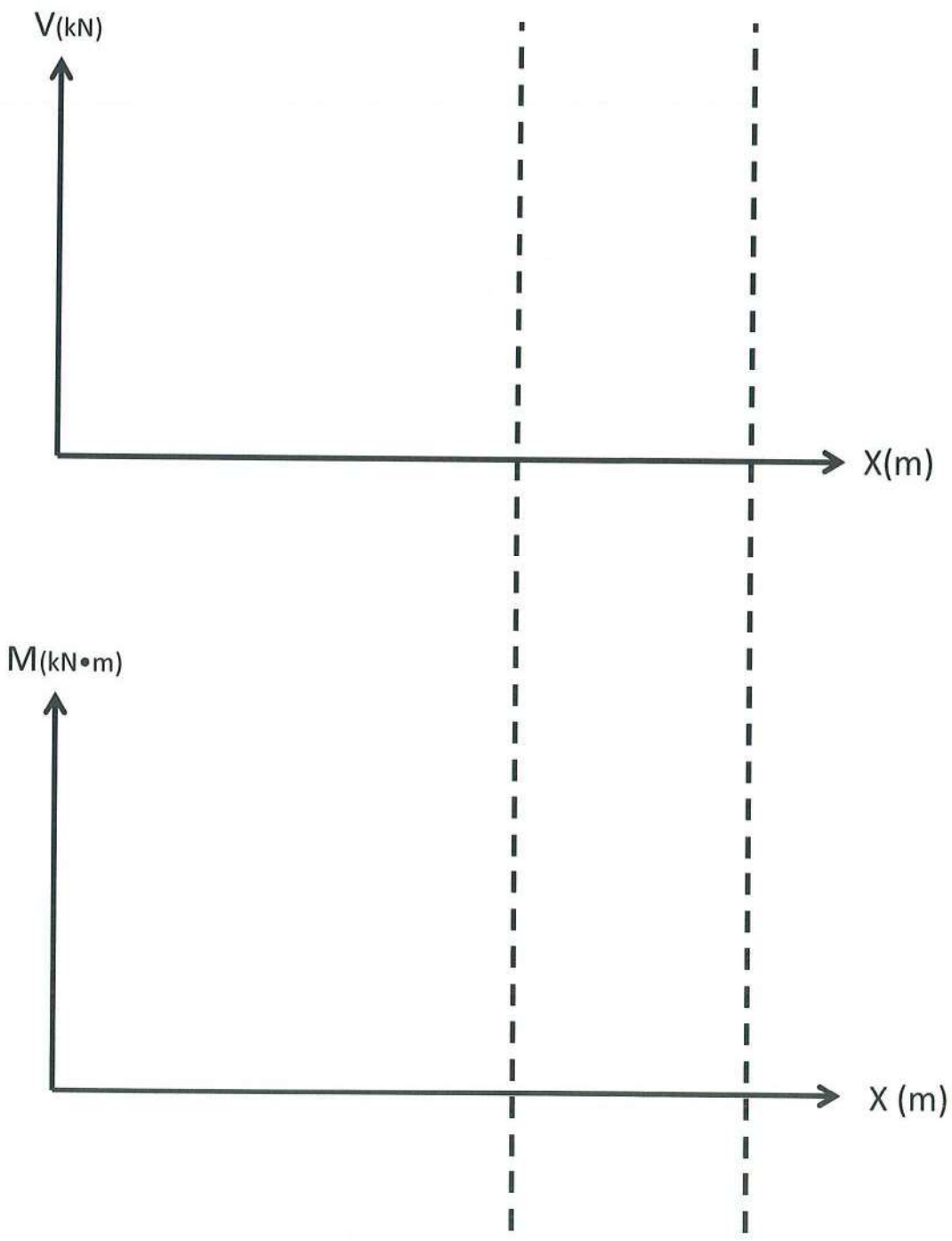
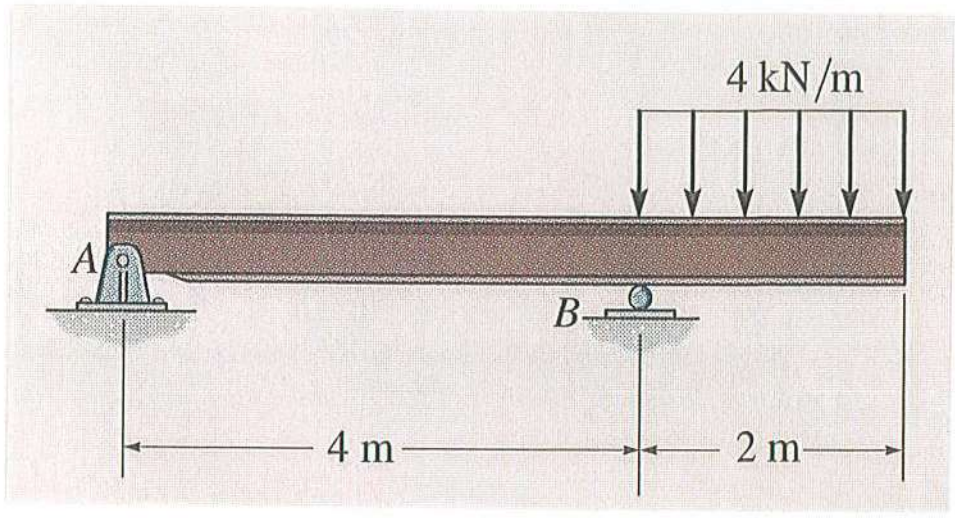
4) _____

Given: Cantilever beam AB is loaded as shown.

Find: Using the graphical method, sketch the shear-force and bending moment diagrams.

Solution:





ME 270 – Basic Mechanics I – Group Quiz

Your Name: _____ Group Members: 1) _____

Date: _____ Period: _____ 2) _____

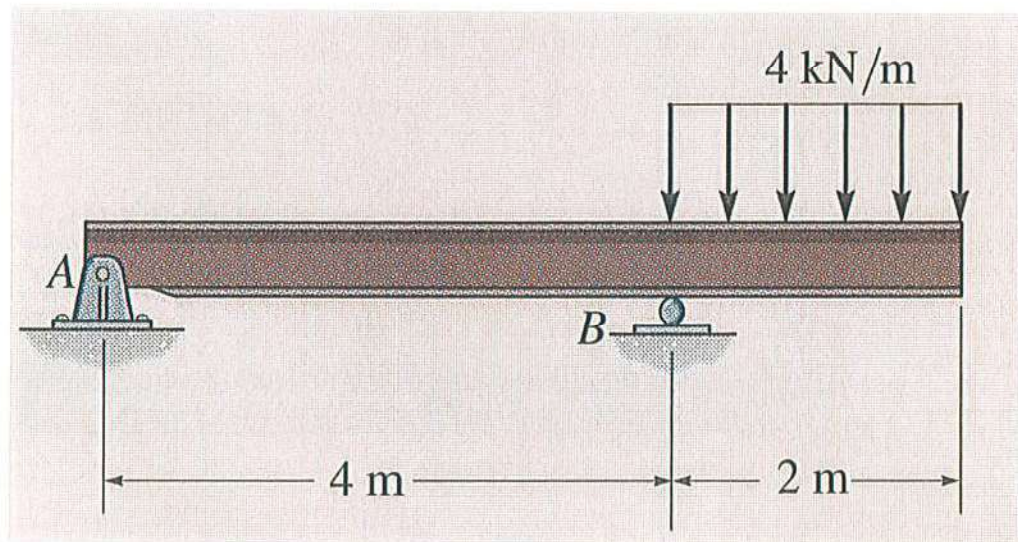
3) _____

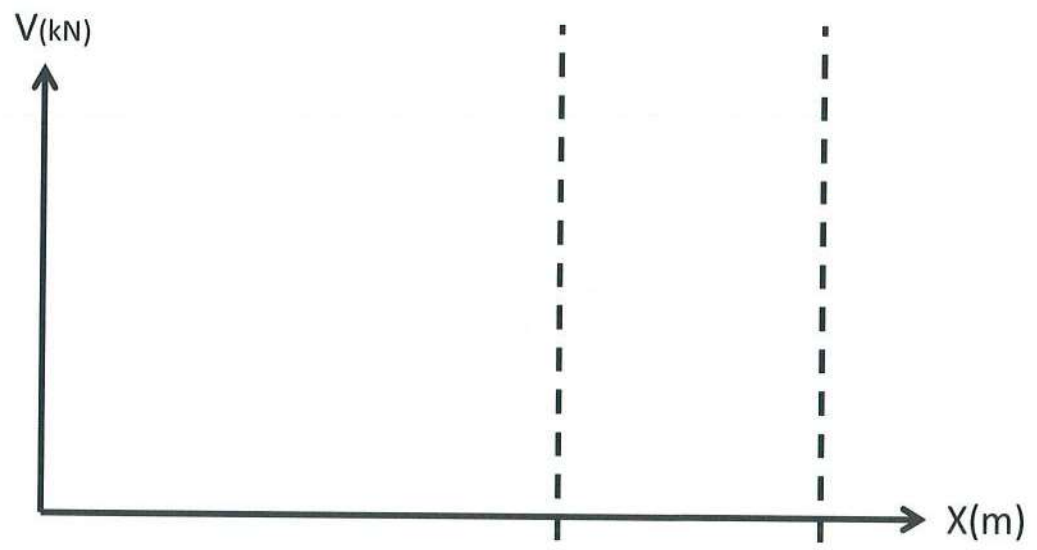
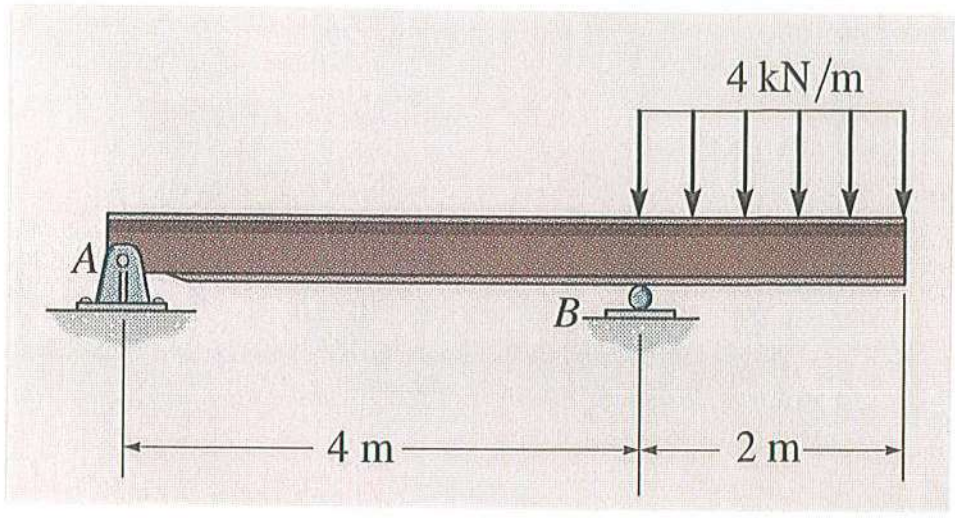
4) _____

Given: Cantilever beam AB is loaded as shown.

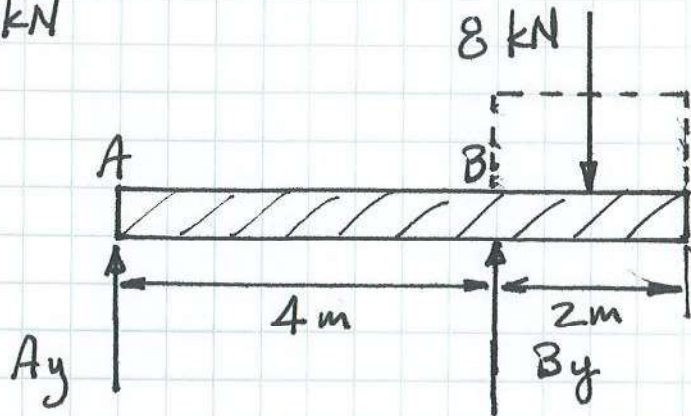
Find: Using the graphical method, sketch the shear-force and bending moment diagrams.

Solution:





$$F_{eq} = \left(4 \frac{\text{kN}}{\text{m}}\right) (2 \text{ m}) = 8 \text{ kN}$$

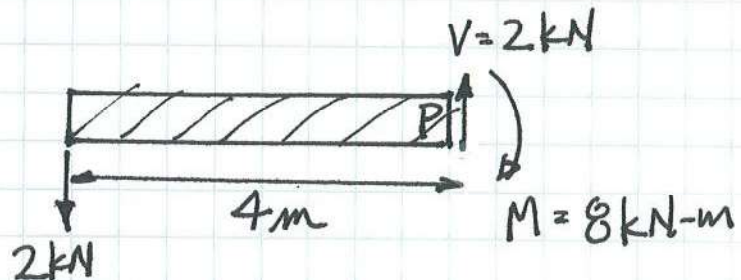


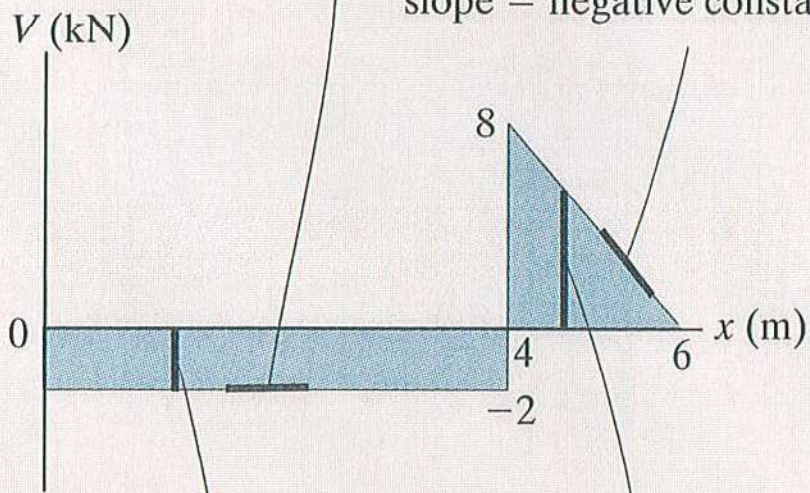
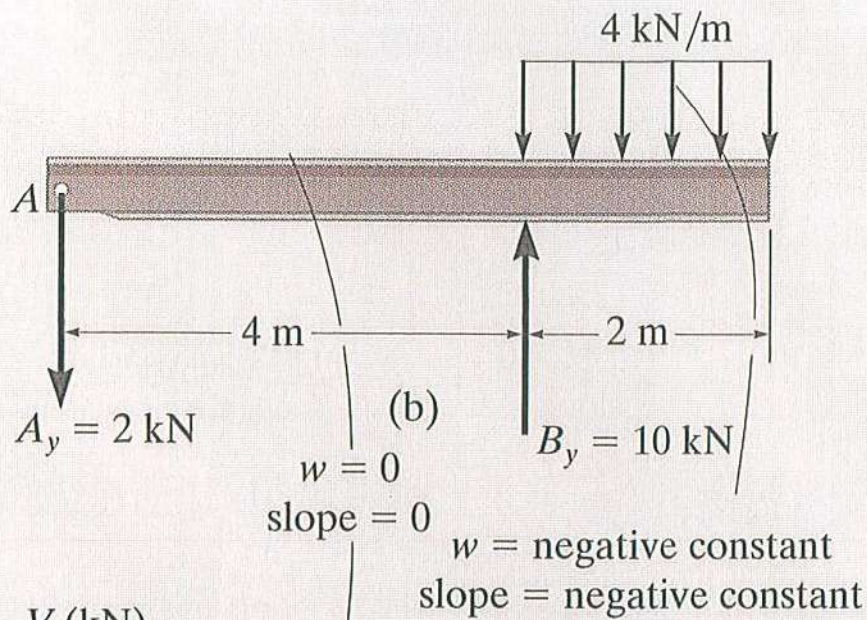
$$\underline{\underline{\sum M_A = 0}} = B_y (4) - 8(5) \Rightarrow \boxed{B_y = 10 \text{ kN}}$$

$$\underline{\underline{\sum F_y = 0}} = A_y + B_y - 8 \Rightarrow \boxed{A_y = -2 \text{ kN}}$$

$$M|_{x=4} = M|_{x=0} + \Delta M = 0 + (-2 \text{ kN})(4 \text{ m})$$

$$\boxed{M|_{x=4} = -8 \text{ kN-m}}$$

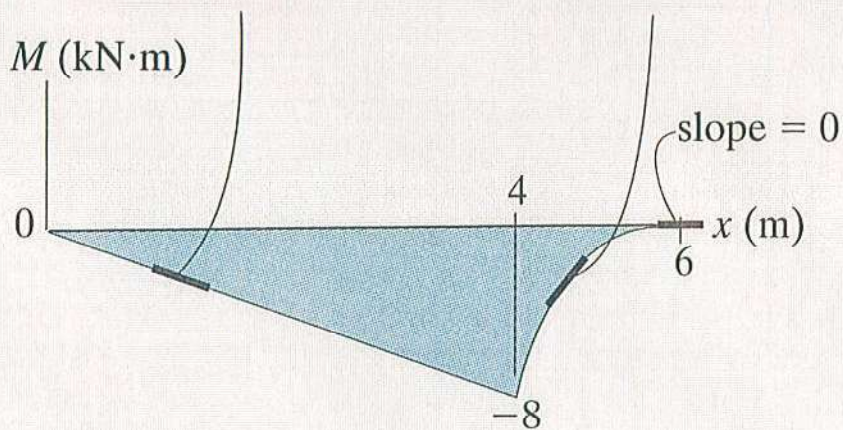




(c)

$V = \text{negative constant}$
slope = negative constant

$V = \text{negative decreasing}$
slope = negative decreasing



(d)