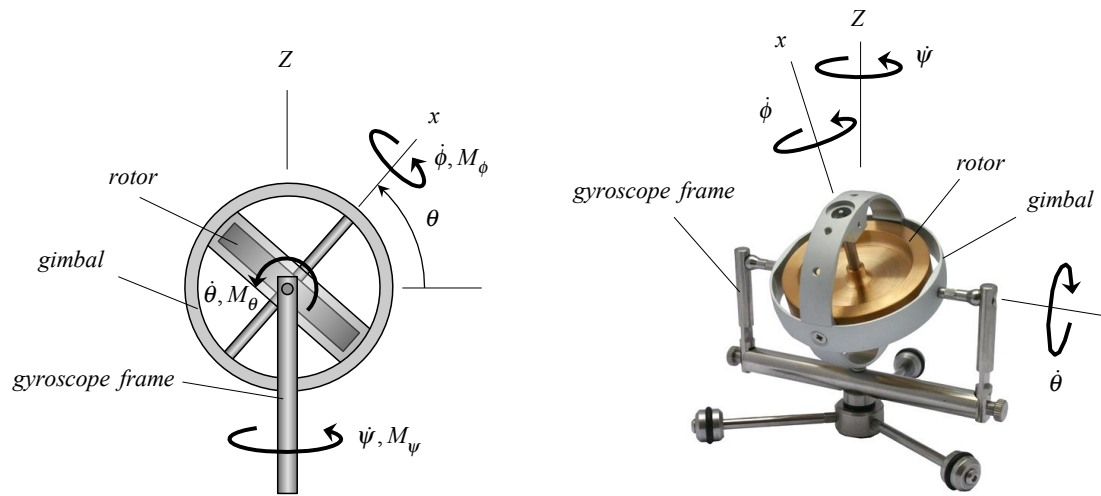
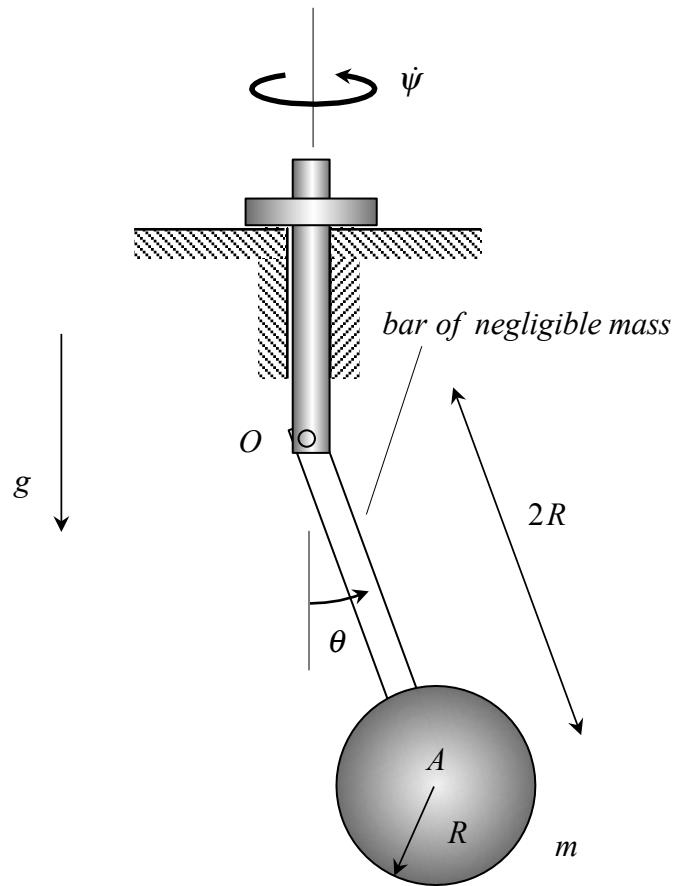


**Problem 9.1 – 25 points**



Consider the rotor of the above gyroscope to be a thin disk of mass  $m$  and radius  $R$ . Ignore the mass of the gimbal and gyroscope frame. The center of mass of the disk lies on the fixed  $Z$ -axis. At some instant in time when the gyroscope has three non-zero rotation rates  $\dot{\phi}$ ,  $\dot{\theta}$  and  $\dot{\psi}$ , a torque of  $M_\psi$  is applied to the frame. Determine  $\ddot{\theta}$  as a result of this applied torque.

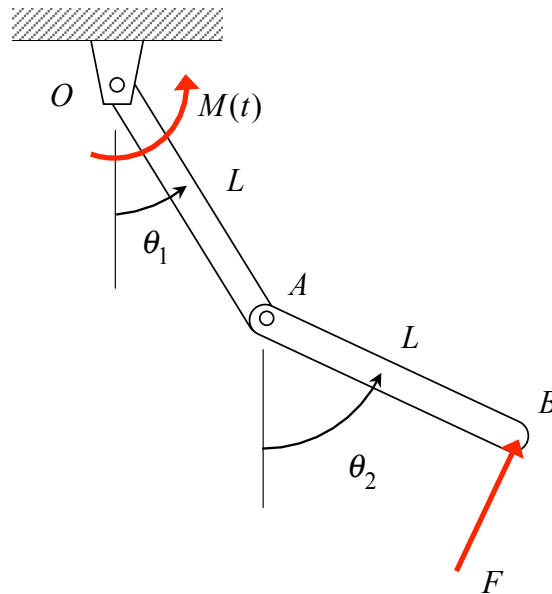
**Problem 9.2 – 25 points**



A homogeneous sphere of mass  $m$  and radius  $R$  is attached to a bar having negligible mass. This bar is pinned to a vertical shaft at end  $O$ , with the shaft rotating about its fixed axis with a constant rate of  $\dot{\psi}$ . Let  $\theta$  represent the angle between  $OA$  and a fixed vertical line.

- Using Lagrange's equations, derive the dynamical equation of motion for the system using the generalized coordinate  $\theta$ .
- From the EOM in a) above, show that the system can precess at a constant rate with  $\theta = \text{constant} = 0$  for all rotation rates. By considering small changes in  $\theta$  from this steady value, evaluate the stability of this steady motion as a function of  $\dot{\psi}$ .
- From the EOM in a) above, show that the system can precess at a constant rate with  $\theta = \text{constant} \neq 0$  for a sufficiently large rotation rate of  $\dot{\psi}$ . What is the critical value of  $\dot{\psi}$  for the appearance of this non-zero precession angle? By considering small changes in  $\theta$  from this steady value, evaluate the stability of this steady motion when it exists.

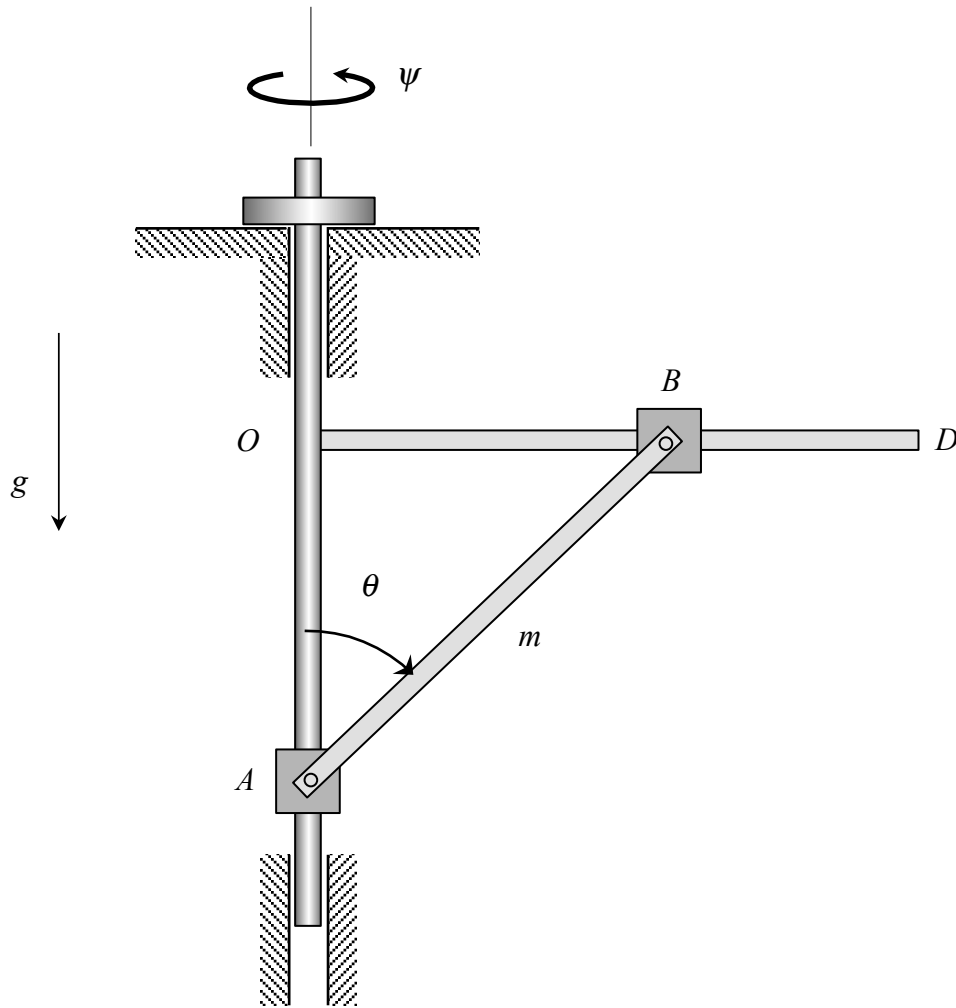
**Problem 9.3 – 25 points**



A two-link pendulum is made of two thin, homogeneous bars, OA and AB, each of mass  $m$  and length  $L$ . A known couple  $M(t)$  acts on bar OA. A force  $F$  acts in a direction that is perpendicular to bar AB, with the magnitude of  $F$  being such that the velocity of end B always collinear with line AB. The absolute angles  $\theta_1$  and  $\theta_2$  are to be used as the generalized coordinates in describing the motion of the system. The system moves in a horizontal plane (i.e., gravity does not affect the motion of the system).

- Use the method of Lagrange multipliers with Lagrange's equations to derive the differential equations of motion for the system.
- How is the Lagrange multiplier in these EOMs related to the applied force  $F$ ?

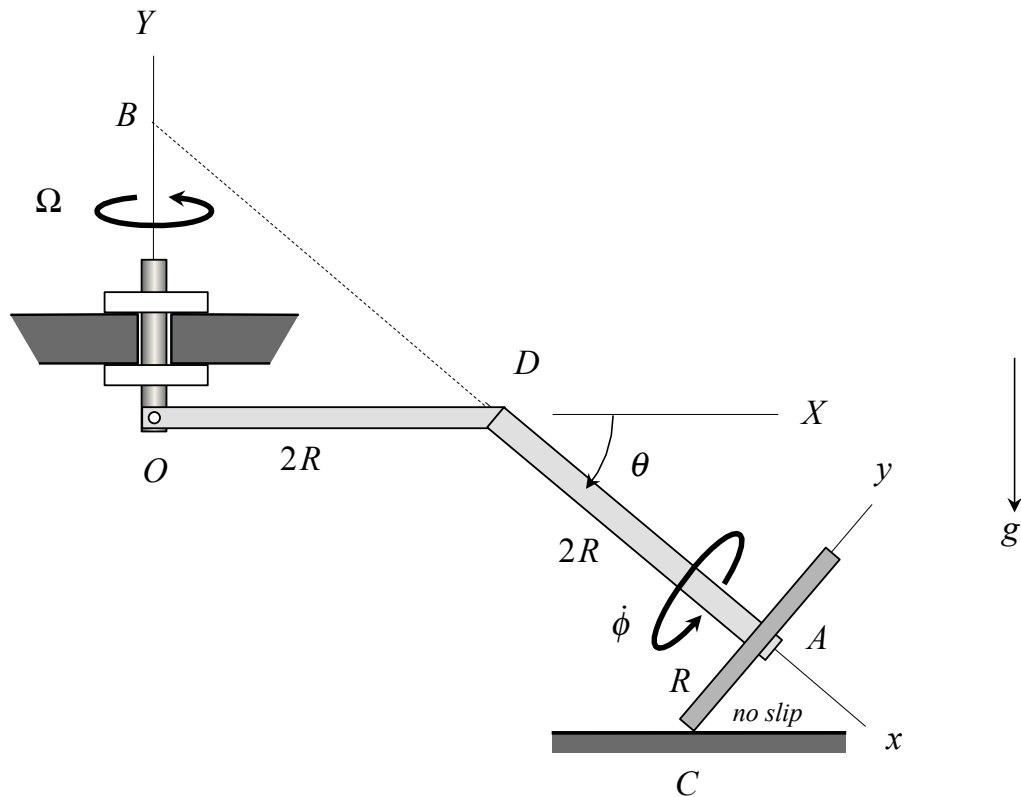
**Problem 9.4 – 25 points**



Arm OD is welded to a thin shaft at a right angle to the fixed rotation axis of the shaft, with the shaft being able to rotate freely about its fixed vertical axis. Let  $I_O$  represent the mass moment of inertia of the shaft and arm OD about the rotation axis of the shaft. Arm AB (of mass  $m$  and length  $L$ ) is free to slide along the vertical shaft at end A and along arm OD at end B. Consider the mass of sliders A and B to be negligible. Let  $\theta$  represent the angle that bar AB makes with the rotation axis of the shaft, and let  $\psi$  be the angle of rotation of the shaft. The system is given initial conditions of:  $\psi(0) = \dot{\psi}(0) = 0$ ,  $\dot{\psi}(0) = \Omega$  and  $\theta(0) = \theta_0$ .

- Use Lagrange's equations to derive the differential equations of motion for the system using the generalized coordinates of  $\theta$  and  $\psi$ .
- Which, if any, of the coordinates in this problem are ignorable? What is the generalized momentum of the system corresponding to the ignorable coordinate (if one exists)?

**Problem 9.5**



A thin disk of mass  $m$  rolls without slipping on a flat, horizontal surface as it freely rotates about the bent arm, as shown, with the arm having negligible mass. The arm, in turn, is pinned to a shaft at  $O$  with the shaft being constrained to rotate about its vertical axis with a constant speed of  $\Omega$ . Determine the normal force acting on the disk by the horizontal surface on which it rolls.