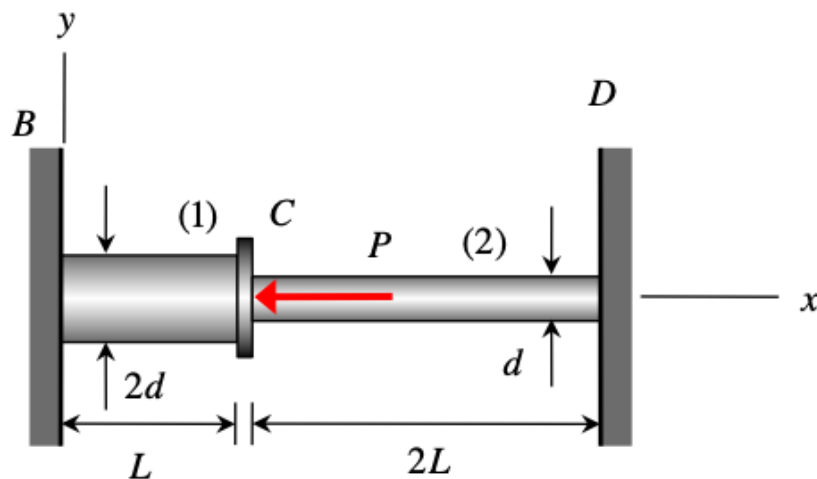


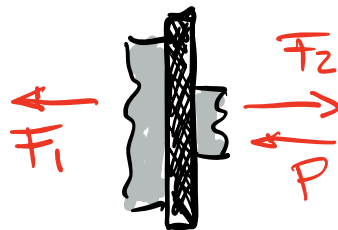
**Problem 3.1 (10 points)**

Two circular cross-section rod elements (1) and (2) are fixed to rigid walls. The rods are made from copper, with Young's modulus  $E$  and coefficient of thermal expansion  $\alpha$ , and they are initially unstressed. An axial load  $P$  is then applied to the rigid connector C, the temperature of element (1) is increased by  $\Delta T$ , and the temperature of element (2) is held constant.

- Assuming all members are under tension, draw the free body diagram of connector C.
- Using the free body diagrams, write down the equations of equilibrium of connector C.
- Is the system statically determinate or indeterminate?
- Write down the force-elongation equation for elements (1) and (2).
- Write down the key compatibility equation(s) that relate the elongation of members (1) and (2).
- Determine the stress in element (1). Indicate if element (1) is under tension or compression.
- Determine the displacement of rigid connector C along the x-axis.



\* Assuming all members under tension.  
\* FBD.



\* Equilibrium:  $\rightarrow \sum F = 0 = F_2 - F_1 - P$  \*1

\* One equation of equilibrium and two unknowns ( $F_1$  and  $F_2$ )  
 $\Rightarrow$  **STATICALLY INDETERMINATE STRUCTURE**

\* Force-elongation equations:

$$e_1 = \frac{F_1 L}{\pi E d^2} + \alpha \Delta T L \quad (*)3$$

$$e_2 = \frac{8 F_2 L}{\pi E d^2} \quad (*)4$$

\* Compatibility conditions

- fixed points:  $u_B = 0$  ;  $u_D = 0$

$$\left. \begin{array}{l} \text{- member \#1 : } e_1 = u_C - u_B \\ \text{- member \#2 : } e_2 = u_D - u_C \end{array} \right\} \Rightarrow e_1 + e_2 = 0 \quad (*)2$$

\* Solve for  $F_1$

$$\text{from } (*)1 \longrightarrow 0 = F_2 - F_1 - P$$

$$\text{from } (*)2, \text{ using } (*)3 \text{ and } (*)4 \longrightarrow \frac{F_1 L}{E \pi d^2} + \alpha \Delta T L + \frac{8 F_2 L}{\pi E d^2} = 0$$

$$\Rightarrow F_1 = - \frac{E \pi d^2}{9} \left[ \frac{8 P}{L E \pi d^2} + \alpha \Delta T \right] \quad (c)$$

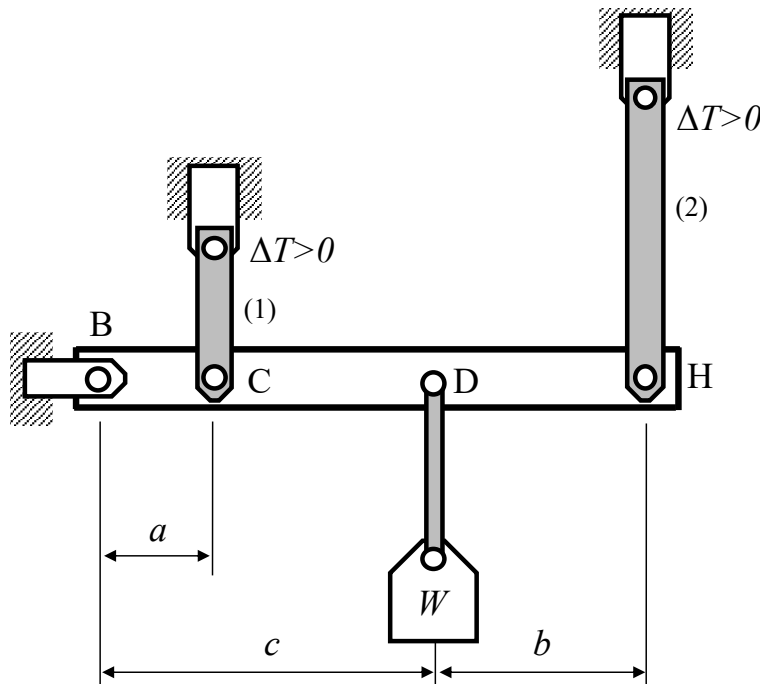
$$\Rightarrow \sigma_1 = \frac{F_1}{A_1} = - \frac{E}{9} \left[ \frac{8 P}{L E \pi d^2} + \alpha \Delta T \right]$$

$$* u_C = e_1 = \frac{F_1 L}{E \pi d^2} + \alpha \Delta T L = \frac{8 L}{9} \left[ \alpha \Delta T - \frac{P}{E \pi d^2} \right]$$

**Problem 3.2 (10 points)**

A rigid bar BCDH is suspended using two deformable rods as shown in the figure. A weight  $W$  of 80 kN is supported from the bar. Both rods undergo the same change in temperature  $\Delta T$ . The coefficient of thermal expansion of steel is  $11 \cdot 10^{-6} \text{ } ^\circ\text{C}^{-1}$  and of bronze is  $19 \cdot 10^{-6} \text{ } ^\circ\text{C}^{-1}$ . Neglecting the weight of the rigid bar,

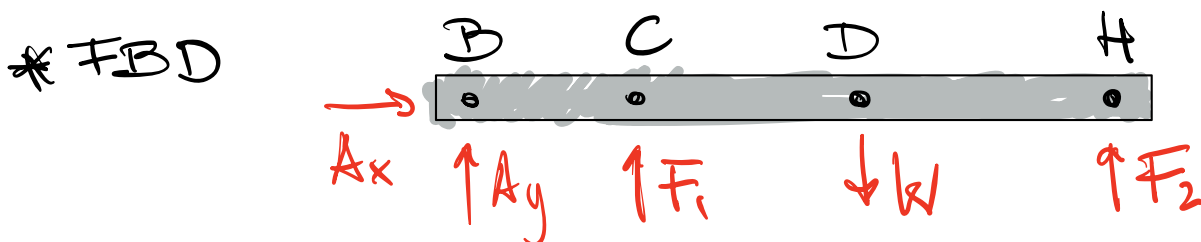
- Assuming that all deformable rods are under tension, draw the free body diagram of the rigid bar BCDH.
- Using the free body diagram, write down the equation(s) of equilibrium of the rigid bar.
- Is the system statically determinate or indeterminate?
- Write down the force-elongation equation for rods (1) and (2).
- Write down the key compatibility equation(s) that relate the elongation of rods (1) and (2).
- Determine the change in temperature that will cause the tensile stress in the steel rod to be equal to 75 MPa.



Steel:  $E_1 = 210 \text{ GPa}$ ,  $L_1 = 1.5 \text{ m}$ ,  $A_1 = 300 \text{ mm}^2$

Bronze:  $E_2 = 80 \text{ GPa}$ ,  $L_2 = 3 \text{ m}$ ,  $A_2 = 1200 \text{ mm}^2$

$a = 1 \text{ m}$ ,  $b = 2 \text{ m}$ ,  $c = 3 \text{ m}$



\* Equilibrium:

$$\sum F_x = 0 \Rightarrow A_x = 0$$

$$\oplus \uparrow \sum F_y = 0 = A_y + F_1 + F_2 - W \quad (*)1$$

$$\oplus \curvearrowright (\sum M)_B = 0 = F_1 \cdot 3W + 5F_2$$

2 equations of equilibrium and 3 unknowns  
 $(\sum F_y = 0; \sum M = 0)$   $(A_y, F_1, F_2)$

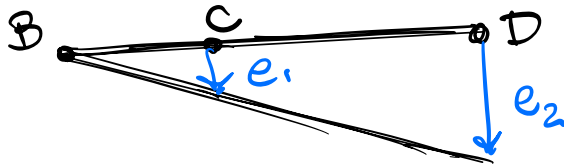
$\Rightarrow$  **STATICALLY INDETERMINATE STRUCTURE**

\* Force-elongation equations

$$e_1 = \frac{F_1 L_1}{E_1 A_1} + \alpha_1 L_1 \Delta T \quad (*)3$$

$$e_2 = \frac{F_2 L_2}{E_2 A_2} + \alpha_2 L_2 \Delta T \quad (*)4$$

\* Compatibility conditions



$$\Rightarrow \frac{e_1}{1m} = \frac{e_2}{5m}$$

$$\Rightarrow e_2 = 5e_1 \quad (*)2$$

\* Solve for  $F_1$

from  $(*)1 \longrightarrow 0 = F_1 - 3W + 5F_2$

from  $(*)2$  using  $(*)3$  and  $(*)4 \longrightarrow \frac{F_2 L_2}{E_2 A_2} + \alpha_2 \Delta T L_2 = 5 \left[ \frac{F_1 L_1}{E_1 A_1} + \alpha_1 \Delta T L_1 \right]$

$$\Rightarrow \left[ \frac{3W - F_1}{5} \right] \frac{L_2}{E_2 A_2} + \alpha_2 \Delta T L_2 = 5 \left[ \frac{F_1 L_1}{E_1 A_1} + \alpha_1 \Delta T L_1 \right]$$

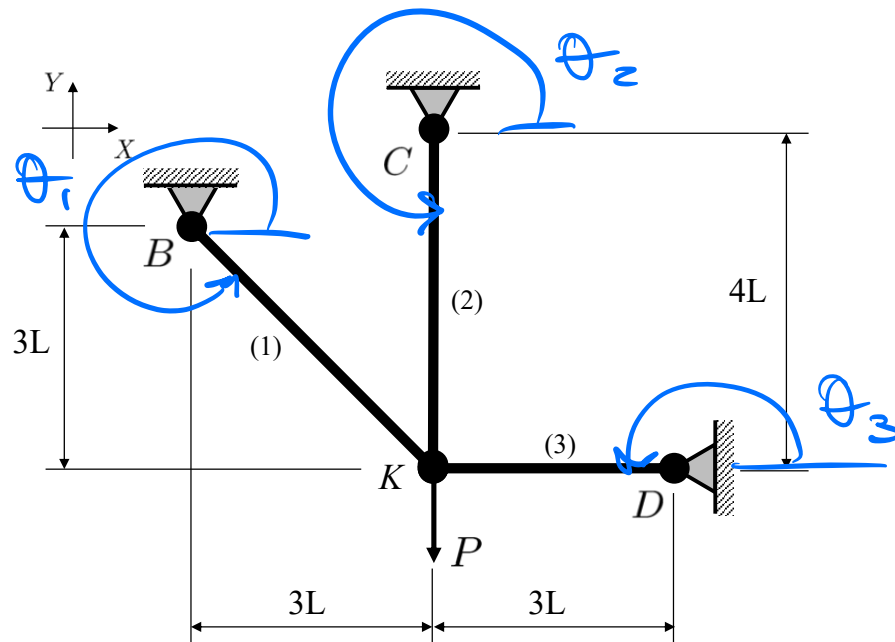
If  $G_1 = \frac{F_1}{A_1} = 75 \text{ MPa}$ , then solve for  $\Delta T$

$$\Rightarrow \Delta T = \frac{\left[ \frac{3W - F_1}{5} \right] \frac{L_2}{E_2 A_2} - 5 \frac{F_1 L_1}{E_1 A_1}}{5 \alpha_1 L_1 - \alpha_2 L_2}$$

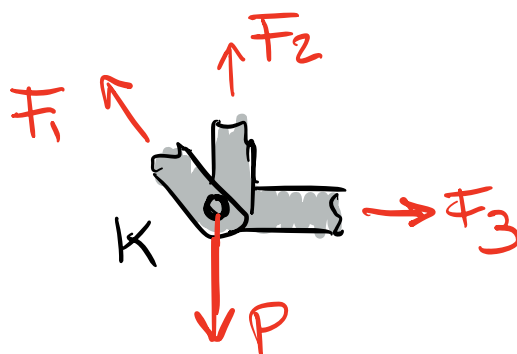
**Problem 3.3 (10 points)**

A planar truss structure is composed of three elastic truss members (1), (2), and (3) of uniform cross-sectional area  $A$ , as shown the figure. Members (1), (2), and (3) are pinned together at K. Members (1) and (2) are composed of an elastic material whose Young's modulus is  $E$ , whereas Young's modulus for (3) is  $2E$ . The structure is acted upon by a concentrated load  $P$  at joint K as shown. Orientation of the x-y axes is indicated.

- Assuming that all deformable rods are under tension, draw the free body diagram of the joint K.
- Using this free body diagram, write down the equilibrium equations of joint K.
- Is the system statically determinate or indeterminate?
- Write down the force-elongation equation for truss members (1), (2) and (3).
- Write down the compatibility equations for the structure. Specifically, write down the equations relating the displacement of joint K in the x and y directions to the axial elongation of each member (i.e.,  $u_K$  and  $v_K$ ).
- Determine the angular orientation  $\theta$  for each member used in the compatibility equations (i.e.,  $\theta_1$ ,  $\theta_2$ , and  $\theta_3$ ).
- Determine the axial load in each truss element. Indicate if the element is under tension or compression.



\* FBD



\* Equilibrium

$$\Rightarrow \sum F_x = 0 = F_3 - F_1/\sqrt{2} \quad (*)1$$

$$\oplus \uparrow \sum F_y = 0 = F_2 + F_1/\sqrt{2} - P \quad (*)2$$

2 equations of equilibrium and 3 unknowns  
( $\sum F_x = 0$ ;  $\sum F_y = 0$ ) ( $F_1, F_2, F_3$ )

$\Rightarrow$  **STATICALLY INDETERMINATE STRUCTURE**

\* Force-elongation equations

$$e_1 = \frac{F_1 (\sqrt{2} \cdot 3L)}{EA} ; \quad e_2 = \frac{F_2 (4L)}{EA} ; \quad e_3 = \frac{F_3 (3L)}{2EA}$$

\* Compatibility conditions

$$e_1 = u_k \cos \theta_1 + v_k \sin \theta_1 = u_k/\sqrt{2} - v_k/\sqrt{2}$$

$$e_2 = u_k \cos \theta_2 + v_k \sin \theta_2 = -v_k$$

$$e_3 = u_k \cos \theta_3 + v_k \sin \theta_3 = -u_k$$

From figure:  $\theta_1 = -45^\circ$ ;  $\theta_2 = -90^\circ$ ;  $\theta_3 = 180^\circ$

$$\Rightarrow e_1 = -e_3/\sqrt{2} + e_2/\sqrt{2} \quad (*)3$$

\* Solve for  $F_1, F_2, F_3$

$$0 = F_3 - F_1/\sqrt{2}$$

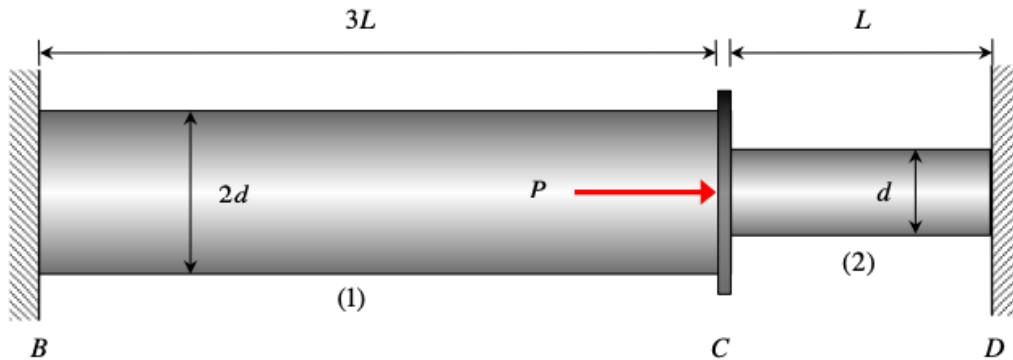
$$0 = F_2 + F_1/\sqrt{2} - P$$

$$\frac{F_1 (\sqrt{2} \cdot 3L)}{EA} = -\frac{1}{\sqrt{2}} \frac{F_3 (3L)}{2EA} + \frac{1}{\sqrt{2}} \frac{F_2 (4L)}{EA}$$

Next, write in terms of  $F_1$  ....



**Problem 3.4 (10 points)**

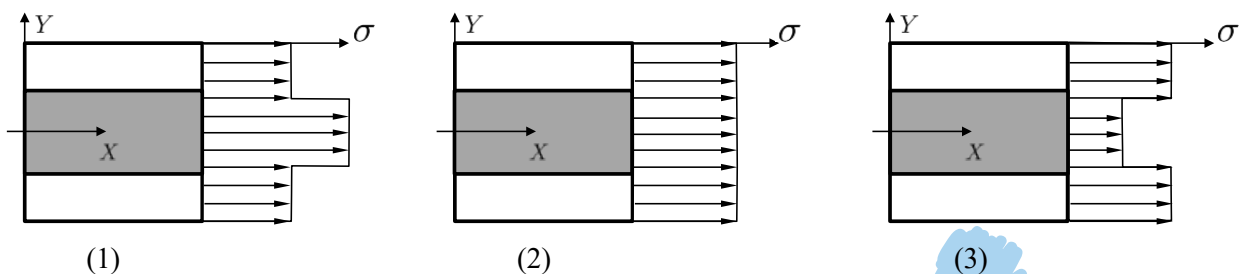
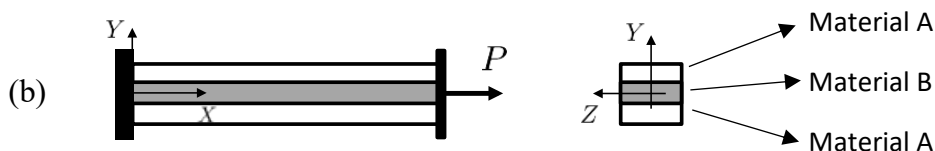


(a) A rod is made up of solid elements (1) and (2) joined by a rigid connector C, with the material of (1) and (2) having the same modulus of elasticity. An axial load  $P$  is applied to C with no thermal loads being present. Let  $F_1$  and  $F_2$  represent the axial loads in elements (1) and (2), respectively. Circle the response below which most accurately describes the relative sizes of  $F_1$  and  $F_2$ :

- i)  $|F_1| > |F_2|$
- ii)  $|F_1| = |F_2|$
- iii)  $|F_1| < |F_2|$
- iv) More information is needed to answer this question.

$$\epsilon_1 + \epsilon_2 = 0 \Rightarrow |\epsilon_1| = |\epsilon_2| \Rightarrow \frac{|F_1| 3L}{E \pi \left(\frac{2d}{2}\right)^2} = \frac{|F_2| L}{E \pi \left(\frac{d}{2}\right)^2} \Rightarrow |F_2| = \frac{3}{4} |F_1|$$

(b) A bimetallic bar with square cross section comprised of two elastic materials is subjected to an axial force  $P$ . Material A, depicted using white, is stiffer than material B, depicted using gray. Specifically, the Young's modulus of material A is two times larger than the Young's modulus of material B, and both materials have the same Poisson's ratio.



$$\epsilon_A = \epsilon_B = \epsilon \Rightarrow \sigma_A = E_A \epsilon ; \sigma_B = E_B \epsilon \Rightarrow \sigma_A > \sigma_B$$

with  $E_A > E_B$