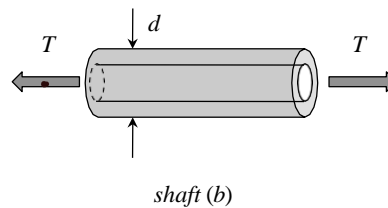
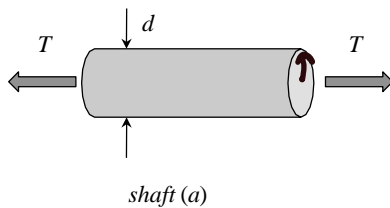


### Conceptual question 8.1

Shaft (a) has a solid cross section with outer radius  $d$ . Shaft (b) has a tubular cross section with an outer radius of  $d$ . Each shaft has the same length and the same shear modulus  $G$ . Let  $\tau_{a,max}$  and  $\tau_{b,max}$  represent the maximum shear stress in shafts (a) and (b), respectively, due to the torque  $T$  applied at the shafts' ends. Circle the correct answer:

- a)  $|\tau_{a,max}| > |\tau_{b,max}|$   
b)  $|\tau_{a,max}| = |\tau_{b,max}|$   
**c)  $|\tau_{a,max}| < |\tau_{b,max}|$**



$$\tau_{max} = \frac{T R}{\boxed{I_p}}$$

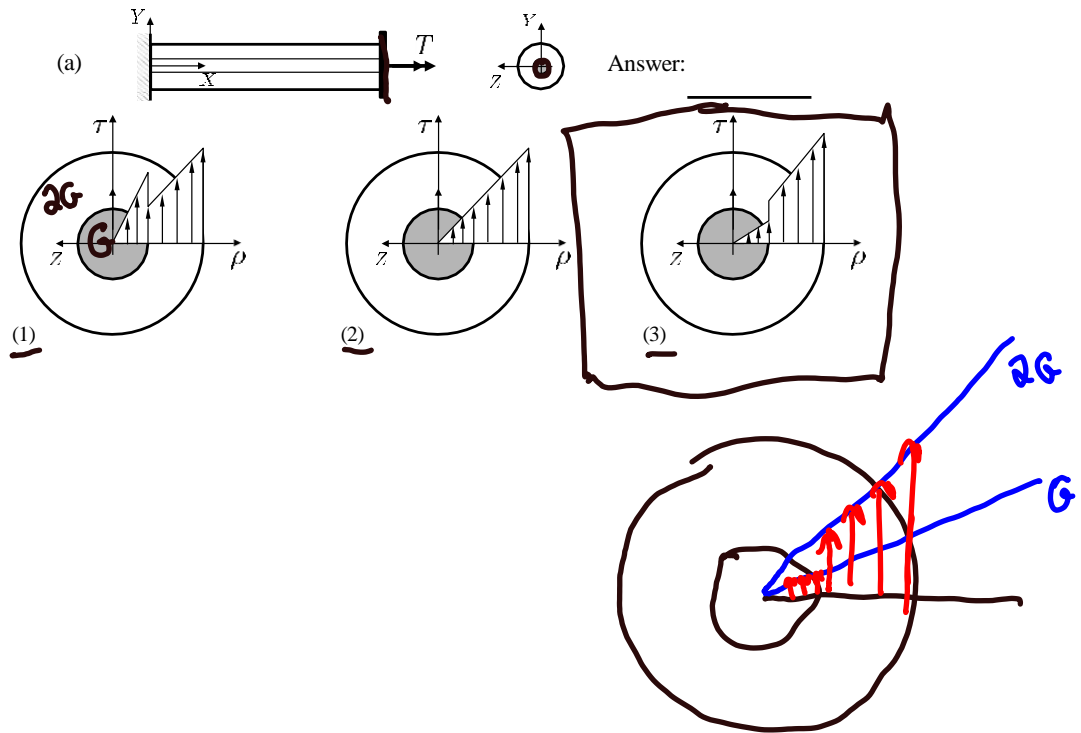
$$I_p(a) = \frac{\pi}{2} R^4$$

$$I_p(b) = \frac{\pi}{2} (R^4 - r_i^4) \leftarrow \text{smaller.}$$

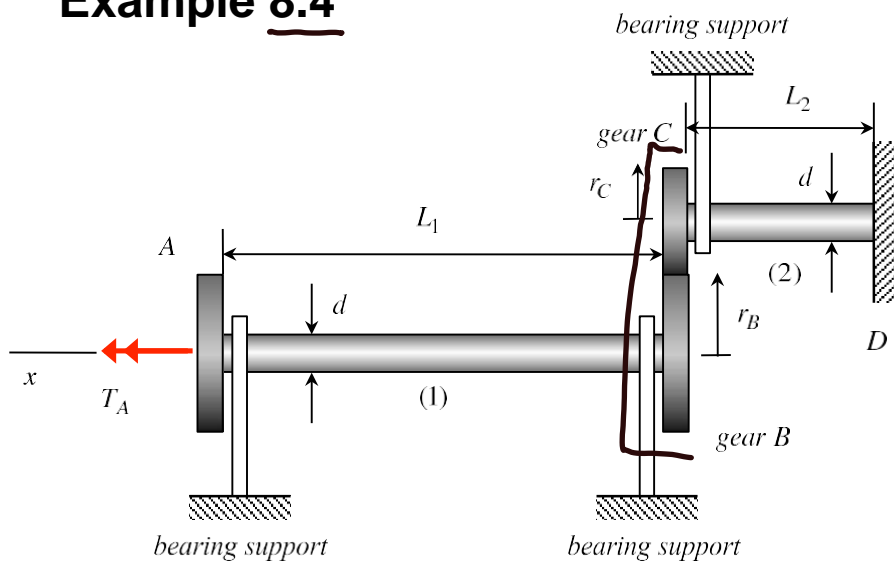
### Conceptual question 8.2

For each loading configuration shown below, indicate the correct stress distribution over a cross section perpendicular to the x-axis.

(a) A bimetallic bar with circular cross section comprised of two elastic materials is subjected to a torque  $T$ . Material A, depicted using white, is stiffer than material B, depicted using gray. Specifically, the Young's modulus of material A is two times larger than the Young's modulus of material B, and both materials have the same Poisson's ratio.

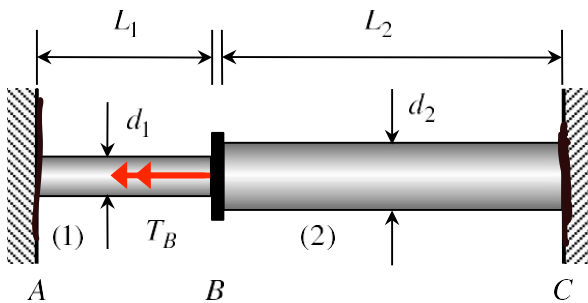


### Example 8.4



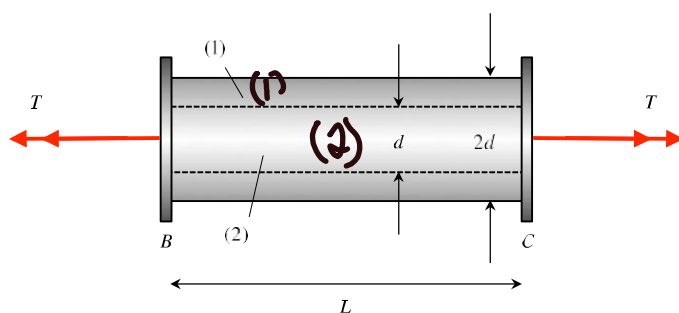
$$\underbrace{-\frac{r_C}{r_B} \phi_C = \phi_B}$$

### Example 8.5



$$\underbrace{\Delta \phi_1 + \Delta \phi_2 = 0}$$

### Example 8.7



$$\underbrace{\Delta \phi_1 = \Delta \phi_2.}$$

Stress analysis of members in torsion

$$\Delta \phi_1 = \frac{T_1 L_1}{G_1 I_{p1}}$$

## Additional Example: Complex assembly

$$\Delta\phi_3 + \Delta\phi_4 = 0$$

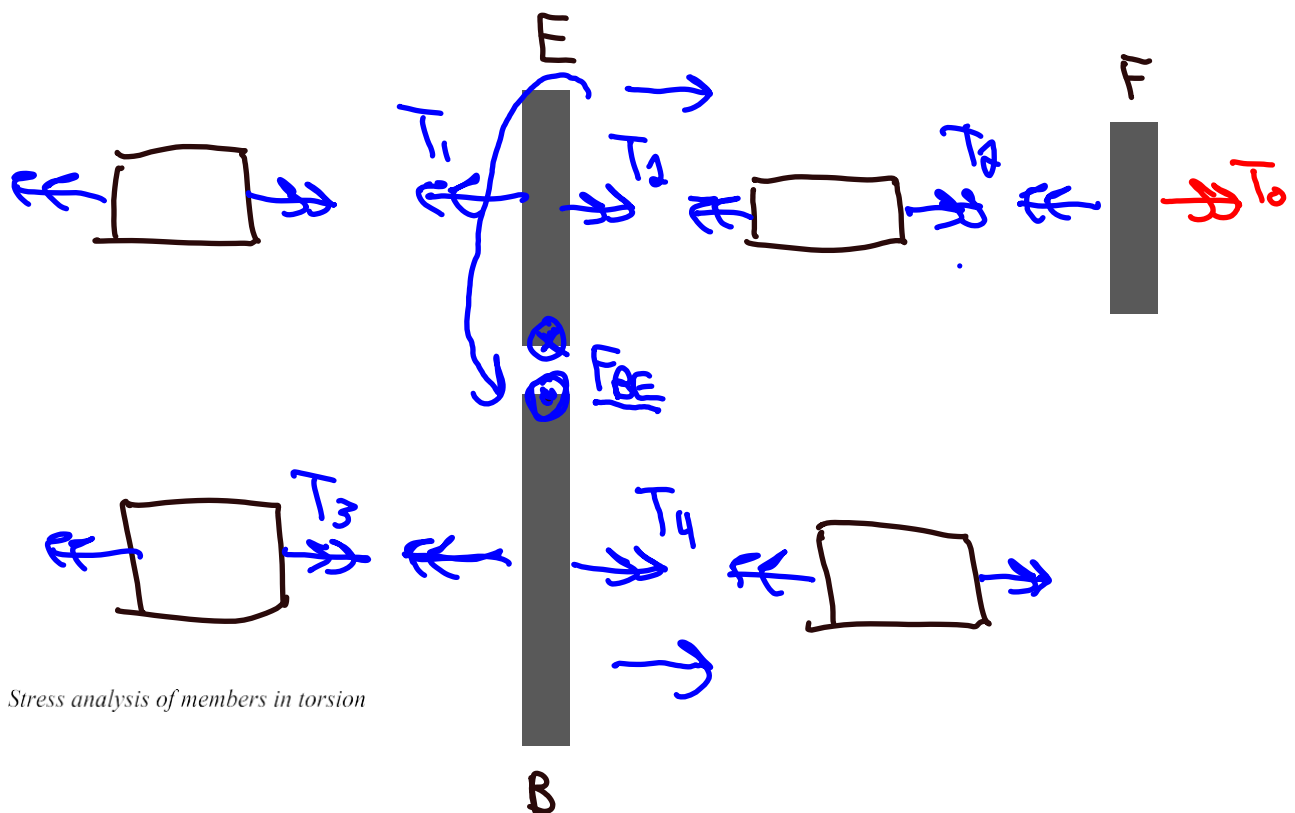
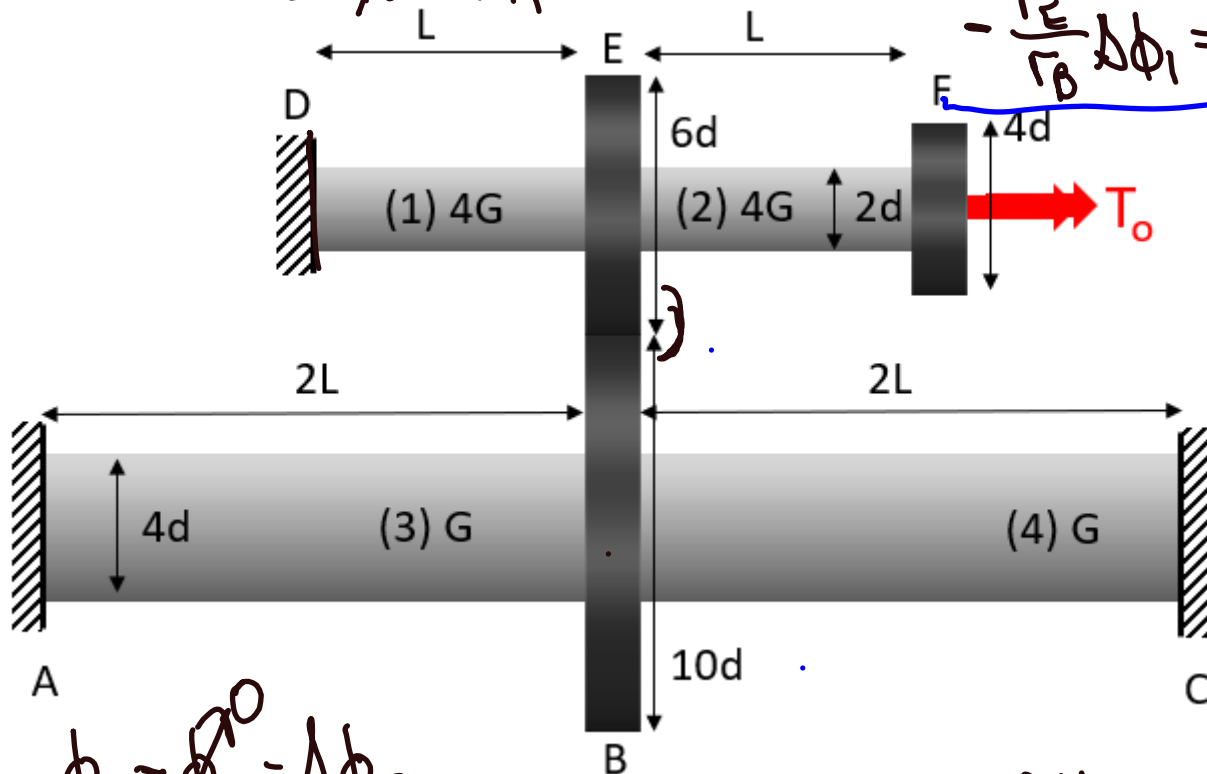
(a) Set up the equilibrium equations.

(b) Identify the key compatibility equations.

$$-\frac{r_E}{r_B} \phi_E = \phi_B$$

$$-\frac{r_E}{r_B} \Delta\phi_1 = \Delta\phi_3$$

$$\phi_E - \phi_D^0 = \Delta\phi_1$$



*Stress analysis of members in torsion*

### 1.) Equilibrium

$$\left. \begin{aligned} (\Sigma M)_F &= T_0 - \underline{T_2} = 0 \\ (\Sigma M)_E &= T_2 - \underline{T_1} + \underline{F_{BE}}(3d) = 0 \\ (\Sigma M)_B &= \underline{T_4} - \underline{T_3} + F_{BE}(5d) = 0 \end{aligned} \right\} \begin{array}{l} 5 \text{ unknowns} \\ 3 \text{ equations.} \\ \hline 2 \text{ compatibility.} \end{array}$$

### 3.) Compatibility

$$\Delta\phi_3 + \Delta\phi_4 = 0$$

$$-\frac{r_E}{r_C} \Delta\phi_1 = \Delta\phi_3$$

### 2.) Torque-Twist.

$$\Delta\phi_1 = \frac{T_1 L_1}{G_1 I_{p1}}$$

$$\Delta\phi_2$$

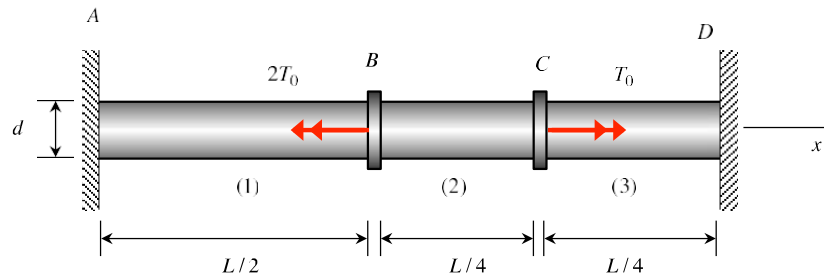
$$\Delta\phi_3$$

$$\Delta\phi_4$$

### Example 8.6

A uniform shaft with fixed ends at A and D is subjected to external torques of magnitudes  $T_0$  and  $2T_0$  as shown in the figure below. The material of the shaft has a shear modulus of  $G$ .

- Determine expressions for the maximum shear stress in each of the three segments of the shaft.
- Determine an expression for the angle of rotation of the shaft at joint B.



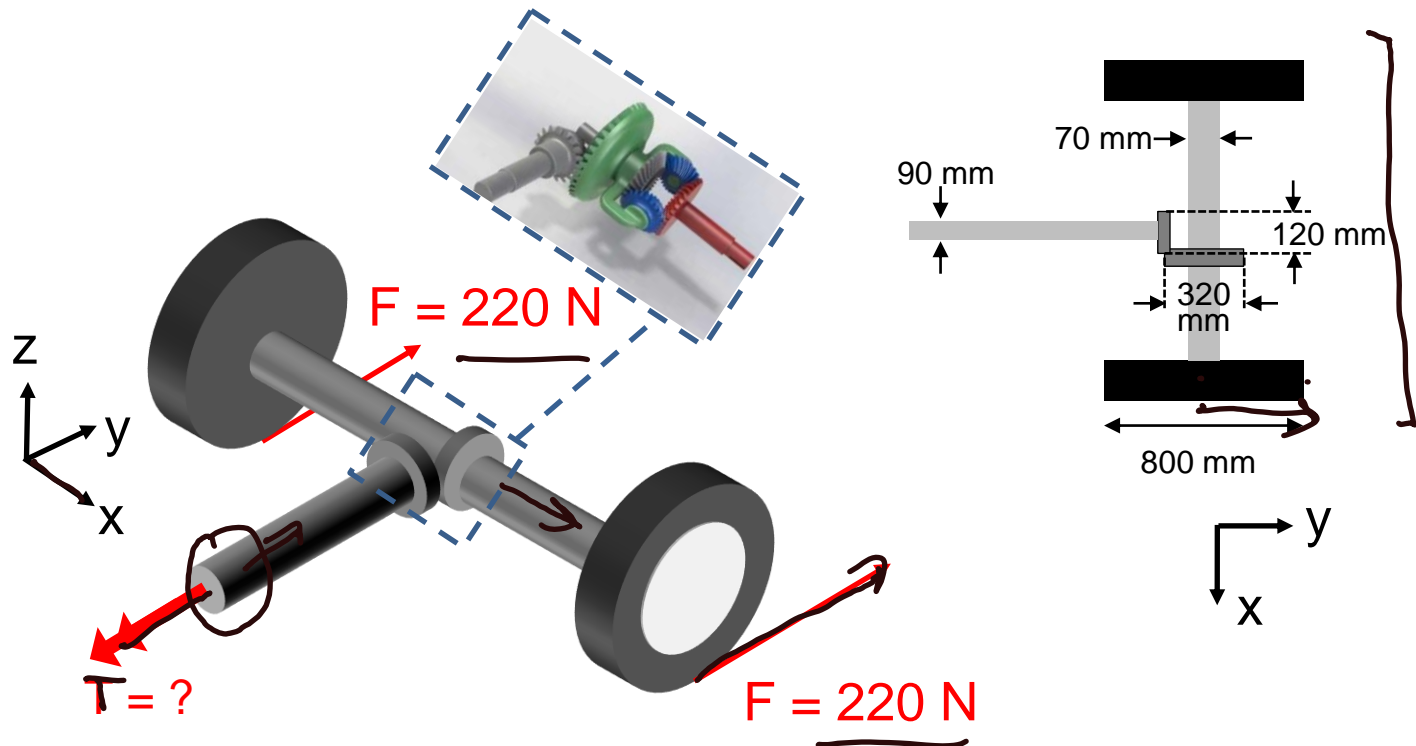


## Motivation for Quiz Format

- Students requested more low-stakes assignments instead of high-stakes ones
- Students learn more from doing things than from watching someone else doing them ←
- Students want to see the relationship to the applications they are interested in



# Lecture 12 Quiz



(a) What torque needs to be applied to the drive shaft for each wheel to output a force of 220 N?

(b) **(completely separate from part a)** A drive shaft must be designed to operate at a torque of 3200 N\*m with an outer diameter of 90 mm and a length of 1320 mm. Using a hollow aluminum tube, what inner diameter is required? The shear modulus of aluminum is 27 GPa. The allowable shear stress in the aluminum is 60 MPa.

