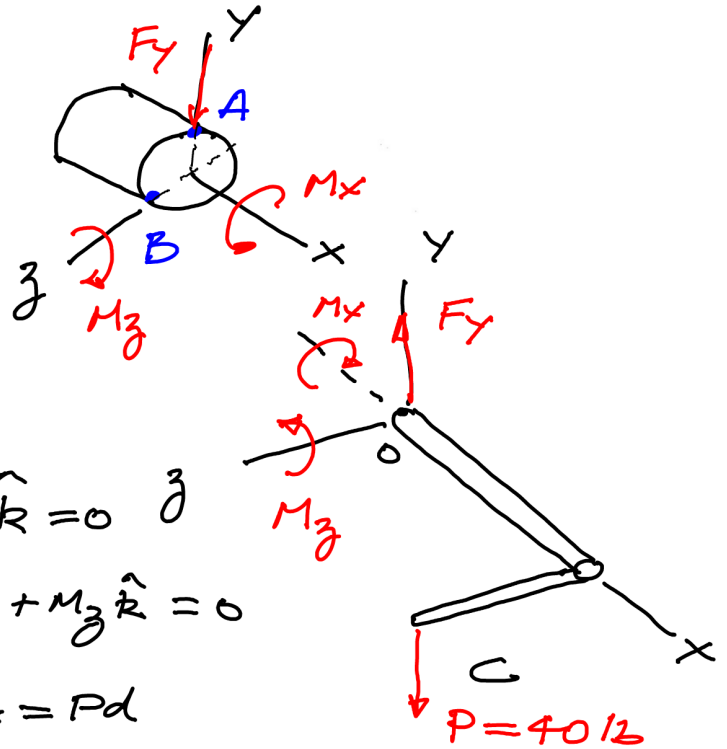
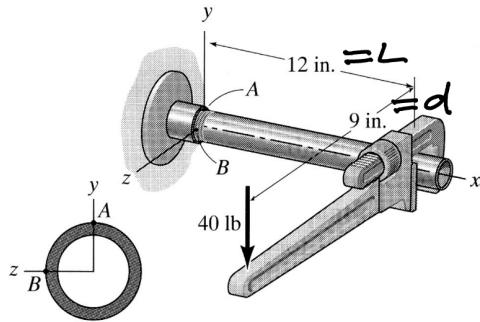


Example 16.4

A vertical force of $P = 40\text{ lb}$ is applied to the end of a pipe wrench, whose handle is parallel to the z -axis. Determine the principal stresses at points A and B on the cross section of the pipe when the pipe threads start. The pipe has an outer diameter of 1 in and a wall thickness of 0.1 in.



Equilibrium

$$\cdot \sum \vec{M}_O = \vec{r}_{C/O} \times \vec{P} - M_x \hat{i} + M_z \hat{k} = 0 \quad \text{}$$

$$\hookrightarrow (L\hat{i} + d\hat{k}) \times (-P\hat{j}) - M_x \hat{i} + M_z \hat{k} = 0$$

$$\hookrightarrow \hat{i}: Pd - M_x = 0 \Rightarrow M_x = Pd$$

$$\hat{k}: -PL + M_z = 0 \Rightarrow M_z = PL$$

$$\cdot \sum F_y = -P + F_y = 0 \Rightarrow F_y = P$$

Point A

$$\tau_{xz} = \frac{M_x r_o}{J}$$

$$\sigma_x = \frac{M_z r_o}{I}$$

$$w/ \quad r_o = 0.5 \text{ in}$$

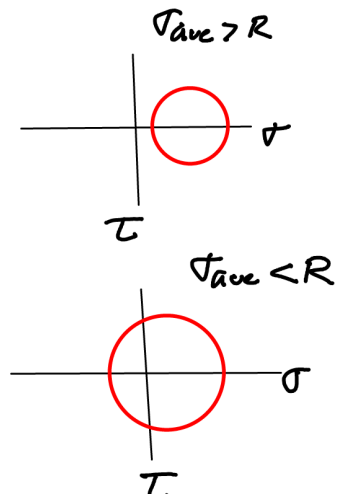
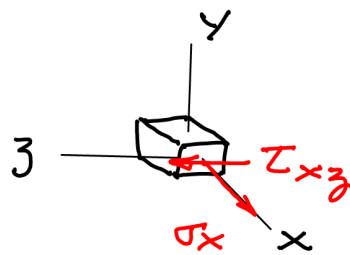
$$J = \frac{\pi}{2} (0.5)^4 - \frac{\pi}{2} (0.4)^4$$

$$I = \frac{\pi}{4} (0.5)^4 - \frac{\pi}{4} (0.4)^4$$

$$\left. \begin{aligned} \sigma_{ave} &= \frac{\sigma_x}{2} \\ R &= \sqrt{\left(\frac{\sigma_x}{2}\right)^2 + \tau_{xz}^2} \end{aligned} \right\}$$

$$\sigma_1 = \sigma_{ave} + R$$

$$\sigma_2 = \sigma_{ave} - R$$



Point B

$$\tau_{xy} = \frac{Mx r_0}{J} + \frac{3}{2} \frac{Fy}{A}$$

$$\text{w/ } A = \pi(0.5)^2 - \pi(0.4)^2$$

$$\begin{cases} \sigma_{ave} = 0 \\ R = \tau_{xy} \end{cases} \left\{ \begin{array}{l} \sigma_1 = \sigma_{ave} + R \\ \sigma_2 = \sigma_{ave} - R \end{array} \right.$$

