

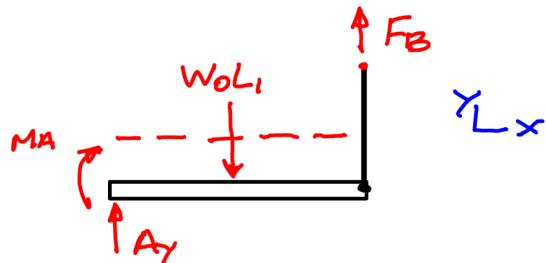
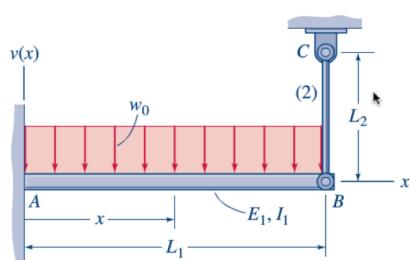
### Example 14.8

For the following examples, set up the problem for determining the requested deflections using Castigliano's method:

- draw appropriate FBDs;
- determine internal results for each section;
- set up the integrals for calculating the required deflections;
- explain how Castigliano's method is used to solve. Discuss the application of dummy forces (when needed) and how to handle redundant forces for indeterminate structures.

#### Problem A

Find the vertical deflection of point B on the beam. Let  $E_2$  and  $A_2$  be the Young's modulus and cross-sectional area, respectively, of member (2).



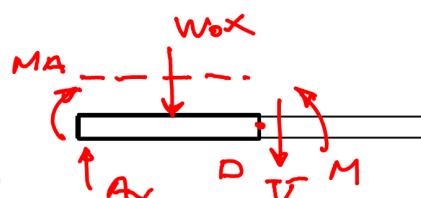
#### External reactions

$$\sum M_A = -M_A - (w_0 L_1) \left(\frac{L_1}{2}\right) + F_B L_1 = 0 \Rightarrow M_A = -\frac{1}{2} w_0 L_1^2 + F_B L_1 \quad (1)$$

$$\sum F_y = A_y - w_0 L_1 + F_B = 0 \Rightarrow A_y = w_0 L_1 - F_B \quad (2)$$

#### Internal reactions

$$\bullet \sum M_D = -M_A - A_y x + w_0 x \left(\frac{x}{2}\right) + M = 0 \quad \hookrightarrow M(x) = M_A + A_y x - \frac{1}{2} w_0 x^2 \quad (3)$$



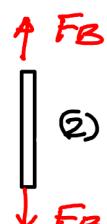
$$\sum F_y = A_y - w_0 x - V = 0$$

$$\hookrightarrow V(x) = A_y - w_0 x \quad (4)$$

- For element (2): See FBD

#### Strain energy ( $\leftrightarrow$ System = (1)+(2))

$$U = \underbrace{\frac{1}{2E_1 I_1} \int_0^{L_1} M(x)^2 dx}_{\text{flexural form (1)}} + \underbrace{\frac{f_s}{2G_1 A_1} \int_0^{L_1} V^2(x) dx}_{\text{shear form (1)}} + \underbrace{\frac{F_B^2 L_2}{2E_2 A_2}}_{\text{axial form (2)}}$$



#### Castigliano's Theorem

$$\delta_C = \frac{\partial U}{\partial F_B} = \frac{1}{E_1 I_1} \int_0^{L_1} M \frac{\partial M}{\partial F_B} dx + \frac{f_s}{G_1 A_1} \int_0^{L_1} V \frac{\partial V}{\partial F_B} dx + \frac{F_B L_2}{E_2 A_2}$$

$$\text{wl} \quad M = M_A + A_y x - \frac{1}{2} w_0 x^2 = -\frac{1}{2} w_0 L_1^2 + F_B L_1 + (w_0 L_1 - F_B)x - \frac{1}{2} w_0 x^2$$

$$V = A_y - w_o x = (w_o L_1 - F_B) x - w_o x$$

$$\therefore \frac{\partial M}{\partial F_B} = L_1 - x$$

$$\frac{\partial V}{\partial F_B} = -x$$

$$\therefore 0 = \frac{1}{E_1 I_1} \int_0^{L_1} [-\frac{1}{2} w_o L_1^2 + F_B L_1 + (w_o L_1 - F_B)x] (L_1 - x) dx \\ + \frac{f_s}{G_1 A_1} \int_0^{L_1} [(w_o L_1 - F_B)x - w_o x] [-x] dx + \frac{F_B L_2}{E_2 A_2}$$

Integrate and solve for  $F_B$ .

Recall  $\delta_B = \frac{F_B L_2}{E_2 A_2}$  = elongation of rod ( $\epsilon$ )

Also:  $V_B = -\delta_B = -\frac{F_B L_2}{E_2 A_2}$