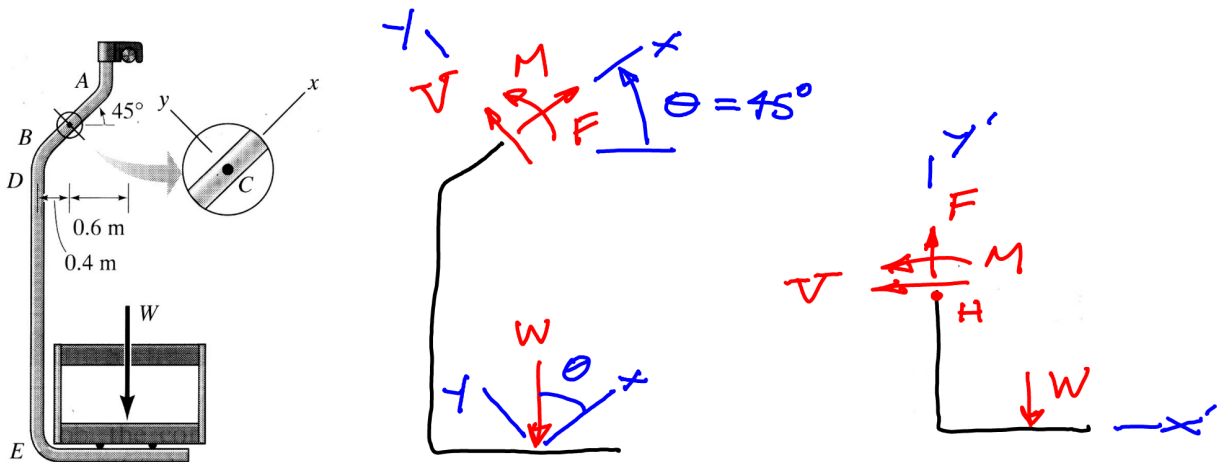


Example 16.6

A chair on a ski lift is supported by a steel pipe with outer and inner diameters of $d_o = 60 \text{ mm}$ and $d_i = 52 \text{ mm}$, respectively. The weight of the pipe may be neglected as compared to the weight $W = 2 \text{ kN}$ of the chair and occupants.

- Determine the stresses at point C on the front section of the pipe at the location shown. The x-axis is parallel to the angled section of the pipe.
- Determine the principal stresses and the maximum in-plane shear stress at point C.
- Determine the maximum tensile stress in the straight section DE of the pipe.



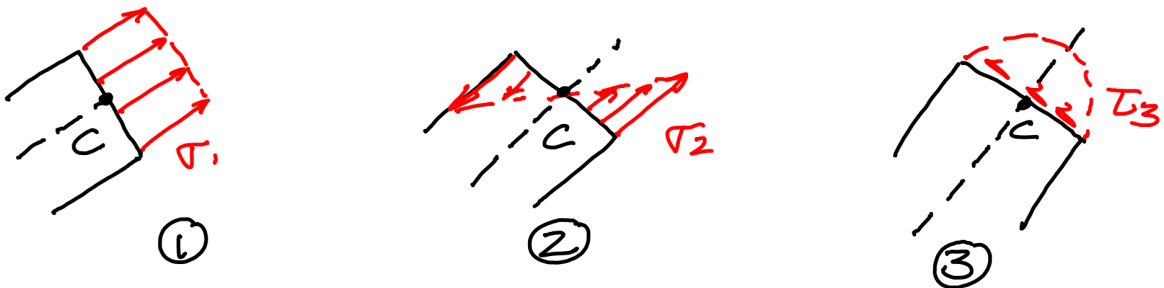
From FBD of cut at C:

$$\sum M_C = -0.6W + M = 0 \quad \Rightarrow M = 0.6W$$

$$\sum F_x = F - W \cos \theta = 0 \Rightarrow F = W \cos \theta$$

$$\sum F_y = V - W \sin \theta = 0 \Rightarrow V = W \sin \theta$$

Stresses at C



At C we have:

$$\tau = \tau_1 = \frac{F}{A}$$

$$\tau = \tau_3|_{\max} = \frac{V A^* \bar{y}^*}{I t}$$

$$w/ \quad A = \pi \left(\frac{d_o}{2}\right)^2 - \pi \left(\frac{d_i}{2}\right)^2$$

$$I = \frac{\pi}{4} \left(\frac{d_o}{2}\right)^4 - \frac{\pi}{4} \left(\frac{d_i}{2}\right)^4$$

$$A^* \bar{y}^* = \frac{\pi}{2} \left(\frac{d_o}{2}\right)^2 \frac{4}{3\pi} \left(\frac{d_o}{2}\right) - \frac{\pi}{2} \left(\frac{d_i}{2}\right)^2 \frac{4}{3\pi} \left(\frac{d_i}{2}\right)$$

$$t = d_o - d_i$$

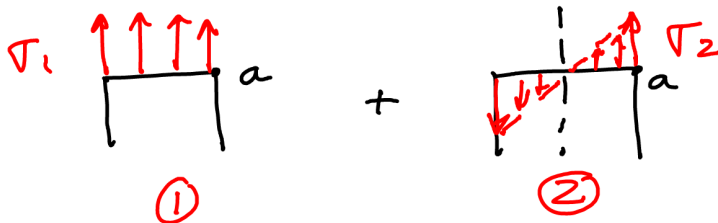
$$\therefore \begin{cases} \sigma_{ave} = \frac{F}{A} \\ R = \sqrt{\left(\frac{F}{A}\right)^2 + \tau^2} \\ \sigma_1 = \sigma_{ave} + R \\ \sigma_2 = \sigma_{ave} - R \\ (\tau_{max})_{in-plane} = R \end{cases}$$

(c) Make cut through section DE

$$\sum F_y = F - W = 0 \Rightarrow F = W$$

$$\sum M_H = -W(1) + M = 0 \Rightarrow M = W$$

Stresses at cut



Make tensile stress occurs at "a":

$$\sigma = \sigma_1 + \sigma_2 |_{max}$$

$$= \frac{F}{A} + \frac{M y}{I} \quad ; \quad y = \frac{d_o}{2}$$