

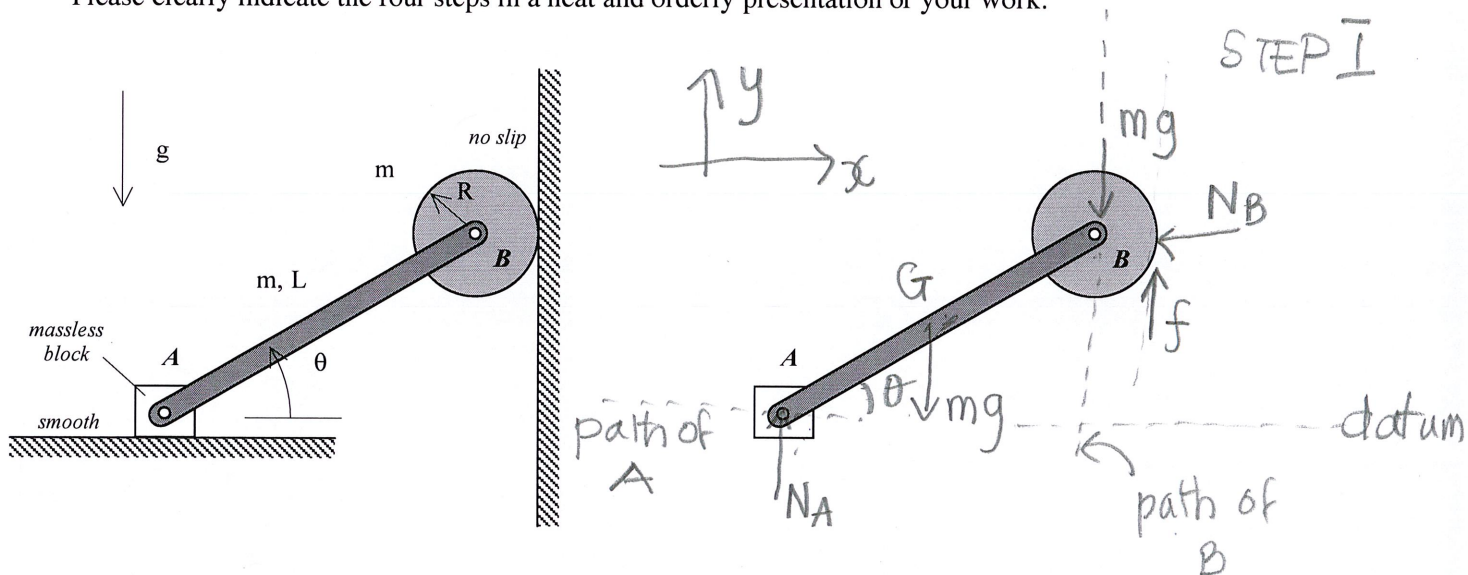
ME 274 – Summer 2010
Examination No. 2
PROBLEM NO. 1

Name _____

Given: System released from rest with $\theta = 36.87^\circ$. Consider the bar and the disk to be homogeneous.

Find: Determine the angular velocity of the bar when $\theta = 0$. Use: $m = 100 \text{ kg}$, $L = 2 \text{ meters}$ and $R = 0.4 \text{ meters}$.

Please clearly indicate the four steps in a neat and orderly presentation of your work.



Step II

Work-Energy

$$0 T_1 + V_1 + \cancel{u_{1 \rightarrow 2}} = T_2 + \cancel{V_2}$$

B and G are at datum in pos (2). $\theta = 0$.

$$mg \frac{L}{2} \sin \theta + \frac{1}{2} L \sin \theta \cdot mg$$

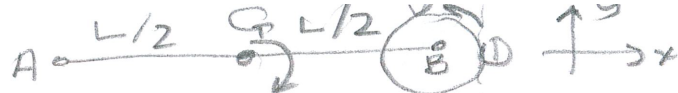
BAR

$$= \frac{1}{2} m v_{G_2}^2 + \frac{1}{2} \frac{I_G}{m} \omega_{BAR_2}^2 + \frac{1}{2} m v_{B_2}^2 + \frac{1}{2} \frac{I_{DISK}}{m} \omega_{DISK_2}^2$$

want this

$$3gL \sin \theta = v_{G_2}^2 + \frac{I_G}{m} \omega_{BAR_2}^2 + v_{B_2}^2 + \frac{I_{DISK}}{m} \omega_{DISK_2}^2 \quad \textcircled{1}$$

Step II



Use kinematics to express v_G , v_{DISK} and v_B in terms of ω_{BAR_2}

BAR pt O to pt. B $\bar{\omega}_{\text{BAR}} = -\omega_{\text{BAR}} \hat{k}$ (assume CW)

$$\bar{v}_B = -v_A \hat{i} + (-\omega_{\text{BAR}}) \hat{k} \times r_{B/A} = -v_A \hat{i} - \omega_{\text{BAR}} L \hat{j}$$

$$-v_B \hat{j} = -v_A \hat{i} - \omega_{\text{BAR}} L \hat{j} \quad \circ \circ \hat{j} \circ$$

$$\underline{v_B = \omega_{\text{BAR}} L} \quad (*)$$

$\uparrow$ down $\uparrow$ CW

$\therefore$ and $\hat{i}$: $\underline{v_A = 0}$ at this instant

DISK pt D to point B

$$\bar{v}_B = \underbrace{\bar{v}_D}_{\text{noslip}} + \underbrace{\bar{\omega}_{\text{DISK}}}_{\omega_{\text{DISK}} \hat{k}} \times \underbrace{r_{B/D}}_{-R \hat{i}} \Rightarrow -v_B \hat{j} = -\omega_{\text{DISK}} R \hat{j}$$

CCW
assumed

$$\therefore \underline{v_B = \omega_{\text{DISK}} \cdot R} \quad (**)$$

$$\underline{\bar{v}_B = -\omega_{\text{DISK}} R \hat{j}} \quad (2)$$

From (*) and (**)

$$\omega_{\text{BAR}} L = \omega_{\text{DISK}} R$$

$$\therefore \underline{\omega_{\text{DISK}} = \omega_{\text{BAR}} \cdot \frac{L}{R}} \quad (2)$$

$$\bar{v}_G = \bar{v}_A + \bar{\omega}_{\text{BAR}} \times r_{G/A} = \vec{0} + -\omega_{\text{BAR}} \hat{k} \times \left(\frac{L}{2} \hat{i}\right)$$

$$\underline{\bar{v}_G = -\frac{L}{2} \omega_{\text{BAR}} \hat{j}} \quad (3)$$

$$\bar{v}_B = -\omega_{\text{BAR}} \frac{L}{R} R \hat{j} = -\omega_{\text{BAR}} L \hat{j} \quad (4)$$

STEP (IV)

Substitute kinematics results into ①

$$3gL \sin \theta = \frac{L^2}{4} \omega_{BAR_2}^2 + \frac{1}{12} L^2 \omega_{BAR_2}^2 + \frac{1}{2} \left(\frac{1}{2} L^2 \right) \left(\frac{\omega_{BAR_2} L}{1} \right)^2$$

$$36 g \cancel{L} \sin \theta = \underbrace{[3L^2 + L^2 + 12L^2 + 6L^2]}_{12L} \omega_{BAR_2}^2$$

$$\bar{\omega}_{BAR} = - \sqrt{\frac{36 g \sin \theta}{12L}} \hat{k} \text{ rad/s}$$

clockwise
motion.

ANS.