

Textbook Reading 5.1-5.9, 6.1, 6.2

You should show all appropriate reservoirs and depict the relevant energy flows (heat transfer and work) between them, through power cycle/refrigeration cycle/heat pump cycle, with correct directions as system diagrams for HW-22 and HW-23.

HW – 22(i)

Consider three different power cycles each receiving 1000 kJ energy input by heat transfer from a reservoir at 500 K. Cycle A rejects 600 kJ by heat transfer to a reservoir at 200 K, cycle B rejects 600 kJ by heat transfer to a reservoir at 300 K, and cycle C rejects 600 kJ by heat transfer to a reservoir at 400 K.

Determine if each power cycle is irreversible or reversible or impossible. Explain why.

HW – 22(ii)

A power cycle receives heat transfer from a hot reservoir and heat transfer at the rate of 10 kW occurs from the power cycle to a reservoir at 300 K. All the power produced by the power cycle is used for operating a reversible heat pump at steady state. The heat pump operates between reservoirs at 270 K and 300 K. There is heat transfer at the rate of 150 kW from the heat pump to the reservoir at 300 K.

- (a) Find the thermal efficiency of the power cycle, in %.
- (b) Determine the required hot reservoir temperature if the power cycle is reversible, in K.

HW – 23(i)

A heat pump cycle operates continuously over a week at steady state to heat a space that is at a constant temperature of 23°C. The heat pump receives energy by heat transfer from a ground source at 5°C while discharging energy by heat transfer to the space at a constant rate of 120,000 kJ/hr. The heat pump consumes 700 kW-hr of electricity over the week and the cost of electricity is \$0.15/kW-hr.

- (a) Determine the total heat transfer into the heat pump from the ground source, in kJ.
- (b) Calculate the coefficient of performance of the heat pump.
- (c) Find the cost of operating the heat pump for one week, in \$.
- (d) Determine the ratio of the actual to maximum coefficient of performance of the heat pump.

HW – 23(ii)

Two irreversible refrigeration cycles operating at steady state are arranged in series. Cycle A removes energy by heat transfer at the rate of 450 W from a cold space at -20°C and has a coefficient of performance of 2.5. Energy by heat transfer rejected from cycle A is supplied to cycle B which operates with a power input of 250 W. Cycle B rejects energy by heat transfer to the environment at 20°C.

- (a) Calculate the power input of cycle A, in W.
- (b) Find the coefficient of performance of cycle B.
- (c) Determine the rate of heat transfer to the environment, in W.

HW – 24(i) (See HW – 8(i))

Complete the following table for propane. You do not need to include system diagram, assumptions, and basic equations for this problem.

State	P , bar	T , °C	v , m ³ /kg	s , kJ/kg-K	x	Phase
a		20			1	
b		20		1.2066		
c	1	40				
d	10		0.0019393			

HW – 24(ii)

Determine whether the following statements are true or false with appropriate justification. You do not need to include system diagram, assumptions, and basic equations for this problem.

- (a) An ideal gas expanding isothermally in a frictionless piston-cylinder system and producing work equal to heat transfer violates the second law of thermodynamics.
- (b) A throttling process is internally reversible.
- (c) A reversible power cycle with a thermal efficiency of 100% violates the first law of thermodynamics.
- (d) A reversible power cycle with a thermal efficiency of 100% violates the second law of thermodynamics.